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May 28, 2018

Numerical methods for nonsmooth mechanical systems Vincent Acary , INRIA Rhône-Alpes, Grenoble.

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# Objectives of the lecture

- Principles and Design of Event-tracking (Event-Driven) schemes. Pros and cons.
- Principles and Design of Event-capturing (Time-stepping) schemes. Pros and cons.
- Comparison between Event-tracking and Event-capturing schemes
- Newmark-type schemes for flexible multibody systems and FEM applications.
- Toward higher order schemes and adaptive time-step strategies

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Event-tracking schemes

L The smooth dynamics and the impact equations

# Nonsmooth Lagrangian Dynamics

## Definition (Nonsmooth Lagrangian Dynamics)

$$\begin{cases} M(q)dv + F(t,q,v^+)dt = di \\ v^+ = \dot{q}^+ \end{cases}$$
(1)

where di is the reaction measure and dt is the Lebesgue measure.

### Decomposition of measure

$$\begin{cases} dv = \gamma dt + (v^{+} - v^{-}) d\nu + dv_{s} \\ di = f dt + p d\nu + di_{s} \end{cases}$$
(2)

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Event-tracking schemes

The smooth dynamics and the impact equations

## Impact equations and Smooth Lagrangian dynamics

Substituting the decomposition of measures into the nonsmooth Lagrangian Dynamics, one obtains

Definition (Impact equations)

$$M(q)(v^+ - v^-)d\nu = pd\nu, \qquad (3)$$

or

$$M(q(t_i))(v^+(t_i) - v^-(t_i)) = p_i,$$
(4)

Definition (Smooth Dynamics between impacts)

$$M(q)\gamma dt + F(t, q, v)dt = fdt$$
(5)

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or

$$M(q)\gamma^{+} + F(t, q, v^{+}) = f^{+} [dt - a.e.]$$
 (6)

Event-tracking schemes

L The smooth dynamics and the impact equations

## The smooth dynamics and the impact equations

#### The impact equations

The impact equations can be written at the time,  $t_i$  of discontinuities:

$$M(q(t_i))(v^+(t_i) - v^-(t_i)) = p_i,$$
(7)

This equation will be solved at the time of impact together with an impact law. That is for an Newton impact law

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L The smooth dynamics and the impact equations

# The smooth dynamics and the impact equations

## The impact equations reduced on the local unknowns

One obtains the following LCP at time  $t_i$  of discontinuities of v:

$$\begin{cases} U_{N}^{+}(t_{i}) = H(q(t_{i}))(M(q(t_{i})))^{-1}H(q(t_{i}))P_{N,i} + U_{N}^{-}(t_{i}) \\ 0 \leq U_{N}^{+}(t_{i}) + eU_{N}^{-}(t_{i}) \perp P_{N,i} \geq 0 \end{cases}$$
(9)

if the matrix  $M(q(t_i))$  is assumed to be invertible.

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L The smooth dynamics and the impact equations

## The smooth dynamics and the impact equations

### The smooth dynamics

The following smooth system are then to be solved (dt - a.e.):

$$\begin{cases} M(q(t))\gamma^{+}(t) + F(t, q, v^{+}) = f^{+}(t) \\ g = g(q(t)) \\ f^{+} = H(q)F^{+}(t) \\ 0 \leq g \perp F^{+}(t) \geq 0 \end{cases}$$
(10)

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Reformulations of the unilateral constraints on Different kinematics levels

# Reformulations of the unilateral constraints on Different kinematics levels

#### Differentiation of the constraints w.r.t time

The constraints g = g(q(t)) can de differentiate with respect to time as follows in the Lagrangian setting:

## Comments. Index reduction techniques.

Solving the smooth dynamics requires that the complementarity condition  $0 \leq g \perp F^+(t) \geq 0$  must be written now at different kinematic level, i.e. in terms of right velocity  $U_{\rm N}^+$  and in terms of accelerations  $\Gamma_{\rm N}^+$ .

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Reformulations of the unilateral constraints on Different kinematics levels

# Reformulations of the unilateral constraints on Different kinematics levels

#### At the velocity level

Assuming that  $U_N^+$  is right-continuous by definition of the right limit of a B.V. function, the complementarity condition implies, in terms of velocity, the following relation,

$$-F^{+} \in \begin{cases} 0 & \text{if } g > 0 \\ 0 & \text{if } g = 0, U_{N}^{+} > 0 \\ ] - \infty, 0] & \text{if } g = 0, U_{N}^{+} = 0 \end{cases}$$
(12)

A rigorous proof of this assertion can be found in (Glocker, 2001).

Event-tracking schemes

Reformulations of the unilateral constraints on Different kinematics levels

# Reformulations of the unilateral constraints on Different kinematics levels

## Equivalent formulations

► Inclusion into  $N_{\rm IR^+}(U_{\rm N}^+)$ 

$$-F^{+} \in \begin{cases} 0 & \text{if } g > 0\\ N_{\mathrm{IR}^{+}}(U_{\mathrm{N}}^{+}) & \text{if } g = 0 \end{cases}$$
(12)

• Inclusion into 
$$N_{T_{\mathrm{IR}^+(g)}}(U_{\mathrm{N}}^+)$$

$$-F^+ \in N_{\mathcal{T}_{\mathrm{IR}^+(g)}}(U^+_{\mathrm{N}})$$
(13)

In a complementarity formalism

$$\begin{array}{ll} \text{if } g = 0 & 0 \leqslant U_{\text{N}}^+ \perp F^+ \geqslant 0 \\ \text{if } g > 0 & F^+ = 0 \end{array} \tag{14}$$

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Reformulations of the unilateral constraints on Different kinematics levels

# Reformulations of the unilateral constraints on Different kinematics levels

#### At the acceleration level

In the same way, the complementarity condition can be written at the acceleration level as follows.

$$-F^{+} \in \begin{cases} 0 & \text{if } g > 0 \\ 0 & \text{if } g = 0, U_{N}^{+} > 0 \\ 0 & \text{if } g = 0, U_{N}^{+} = 0, \Gamma_{N}^{+} > 0 \\ ] - \infty, 0] & \text{if } g = 0, U_{N}^{+} = 0, \Gamma_{N}^{+} = 0 \end{cases}$$
(15)

A rigorous proof of this assertion can be found in (Glocker, 2001).

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Event-tracking schemes

Reformulations of the unilateral constraints on Different kinematics levels

# Reformulations of the unilateral constraints on Different kinematics levels

## Equivalent formulations

► Inclusion into a cone  $N_{\rm IR^+}(\Gamma_{\rm N}^+)$ 

$$-F^{+} \in \begin{cases} 0 & \text{if } g > 0 \\ 0 & \text{if } g = 0, U_{N}^{+} > 0 \\ N_{\mathrm{IR}^{+}}(\Gamma_{N}^{+}) \end{cases}$$
(15)

► Inclusion into 
$$N_{\mathcal{T}_{T_{\mathrm{IR}^+}(g)}(U_{\mathrm{N}}^+)}(\Gamma_n^+)$$
  
-  $F^+ \in N_{\mathcal{T}_{T_{\mathrm{IR}^+}(g)}(U_{\mathrm{N}}^+)}(\Gamma_n^+)$  (16)

In the complementarity formalism,

$$\begin{array}{ll} \text{if } g = 0, \, U_{\text{N}}^{+} = 0 & 0 \leqslant \Gamma_{\text{N}}^{+} \perp F^{+} \geqslant 0 \\ \text{otherwise} & F^{+} = 0 \end{array} \tag{17}$$

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Reformulations of the unilateral constraints on Different kinematics levels

# Reformulations of the unilateral constraints on Different kinematics levels

### Trivial inclusions

$$N_{\mathcal{K}}(g(q)) \supset N_{\mathcal{T}_{\mathrm{IR}^+}(g(q))}(U_{\mathrm{N}}^+) \supset N_{\mathcal{T}_{\mathrm{IR}^+}(g(q))}(U_{\mathrm{N}}^+)}(\Gamma_n^+)$$
(18)

Event-tracking schemes

- Reformulations of the smooth dynamics at acceleration level.

Reformulations of the smooth dynamics at acceleration level.

The smooth dynamics as an inclusion

Event-tracking schemes

Reformulations of the smooth dynamics at acceleration level.

### Reformulations of the smooth dynamics at acceleration level.

#### The smooth dynamics as a LCP

When the condition, g = 0,  $U_N^+ = 0$  is satisfied, we obtain the following LCP

$$\begin{cases} \mathcal{M}(q(t))\gamma^{+}(t) + F(t, q, v^{+}) = \nabla_{q}g(q(t))F^{+}(t) \\ \Gamma_{N}^{+} = \nabla_{q}g^{T}(q)\gamma^{+} + \frac{d}{dt}(\nabla_{q}g^{T}(q))v^{+} \\ 0 \leqslant \Gamma_{N}^{+} \perp F^{+} \geqslant 0 \end{cases}$$
(20)

which can be reduced on variable  $\Gamma_N^+$  and  $F^+$ , if M(q(t)) is invertible,

$$\begin{cases} \Gamma_{N}^{+} = \nabla_{q}g^{T}(q)M^{-1}(q(t))(-F(t,q,v^{+})) + \frac{d}{dt}(\nabla_{q}g^{T}(q))v^{+} \\ + \nabla_{q}g(q)M^{-1}\nabla_{q}g(q(t))F^{+}(t) \\ 0 \leqslant \Gamma_{N}^{+} \perp F^{+} \geqslant 0 \end{cases}$$
(21)

L The case of a single contact.

# The case of a single contact.

#### Two modes for the nonsmooth dynamics

1. The constraint is not active. 
$$F^+ = 0$$

$$M(q)\gamma^{+} + F(\cdot, q, \nu) = 0$$
<sup>(22)</sup>

In this case, we associate to this step an integer,  $status_k = 0$ .

2. The constraint is active. Bilateral constraint  $\Gamma_N^+ = 0$ ,

$$\begin{bmatrix} M(q) & -\nabla_q g(q) \\ \nabla_q g^T(q) & 0 \end{bmatrix} \begin{bmatrix} \gamma^+ \\ F^+ \end{bmatrix} = \begin{bmatrix} -F(\cdot, q, v) \\ \nabla_q g^T(q) v^+ \end{bmatrix}$$
(23)

In this case, we associate to this step an integer,  $status_k = 1$ .

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# The case of a single contact.

[Case 1]  $status_k = 0$ . Integrate the system (22) on the time interval  $[t_k, t_{k+1}]$ -25pt-Case 1.1  $g_{k+1} > 0$ . The constraint is still not active  $status_{k+1} \leftarrow 0$ Case 1.2  $g_{k+1} = 0, U_{N,k+1} < 0$ An impact occurs Solve the impact equation (9) with  $U^- \leftarrow U_{N,k+1} < 0$  $U_{\mathbb{N},k+1} \leftarrow U^+$ . Two cases are then possible: Case 1.2.1  $U_+ > 0.$  25pt The constraint ceases to be active status<sub>k+1</sub>  $\leftarrow 0$ . Case 1.2.2  $U_+ = 0$ . The relative post-impact velocity vanishes Solve the LCP (20) to obtain the new status. Three cases are then possible: Case 1.2.2.1  $\Gamma_{N,k+1} > 0, F_{k+1} = 0$ The constraint is still not active  $status_{k+1} \leftarrow 0.$ Case 1.2.2.2  $\Gamma_{N,k+1} = 0, F_{k+1} > 0$ The constraint has to be activated  $status_{k+1} \leftarrow 1$ . Case 1.2.2.3  $\Gamma_{N,k+1} = 0, F_{k+1} = 0$ This case is undetermined. We need to know the value of  $\dot{\Gamma}_{\rm N}^+$ .

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Event-tracking schemes

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# The case of a single contact.

[Case 1]  $status_k = 0$ . Integrate the system (22) on the time interval  $[t_k, t_{k+1}]$ -25pt-Case 1.3  $g_{k+1} = 0, U_{N k+1} = 0$ we have grazing constraint Solve the LCP (20) to obtain the new status assuming that  $U^+ = U^- = U_{{
m N},k+1}$  . Three cases are then possible: Case 1.3.1  $\Gamma_{N,k+1} > 0, F_{k+1} = 0$ The constraint is still not active  $status_{k+1} \leftarrow 0.$ Case 1.3.2  $\Gamma_{N,k+1} = 0, F_{k+1} > 0$ The constraint has to be activated status<sub>k+1</sub>  $\leftarrow$  1. Case 1.3.3  $\Gamma_{N,k+1} = 0, F_{k+1} = 0$ This case is undetermined. We need to know the value of  $\dot{\Gamma}_{N}^{+}$ .  $c_{5,k} = 1.4 \ g_{k+1} = 0, U_{N,k+1} > 0$ Activation of constraints not detected. Seek for the first time  $t_*$  such that  $g(q(t_*)) = 0$ .  $t_{k+1} \leftarrow t_*$ . Perform all of this procedure keeping with  $status_k \leftarrow 0$ . Case 1.5  $g_{k+1} < 0$ Activation of constraints not detected. Seek for the first time  $t_*$  such that  $g(q(t_*)) = 0$ .  $t_{k+1} \leftarrow t_*$ . Perform all of this procedure keeping with  $status_k \leftarrow 0$ . ◆ロ → ◆御 → ◆臣 → ◆臣 → ○ ● ● ● ● ●

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The case of a single contact.

# The case of a single contact.

[Case 2]  $status_k = 1$ Integrate the system (23) on the time interval  $[t_k, t_{k+1}]$ -25pt-Case 2.1  $g_{k+1} \neq 0$  or  $U_{N,k+1} = 0$ Something is wrong in the time integration or the drift from the constraints is too huge. Case 2.2  $g_{k+1} = 0, U_{N,k+1} = 0$ In this case, we assume that  $U^+ = U^- = U_{N,k+1}$  and we compute  $\Gamma_{N,k+1}, F_{k+1}$ thanks to the LCP (20) assuming that  $U^+ = U^- = U_{N,k+1}$ . Three cases are then possible 25pt Case 2.2.1  $\Gamma_{N k+1} = 0, F_{k+1} > 0$ The constraint is still active. We set  $status_{k+1} = 1$ . Case 2.2.2  $\Gamma_{N,k+1} > 0, F_{k+1} = 0$ The bilateral constraint is no longer valid. We seek for the time  $t_*$  such that  $F^+ = 0$ . We set  $t_{k+1} = t_*$  and we perform the integration up to this instant. We perform all of these procedure at this new time  $t_{k+1}$ Case 2.2.3  $\Gamma_{N,k+1} = 0, F_{k+1} = 0$ This case is undetermined. We need to know the value of  $\dot{\Gamma}_{N}^{+}$ .

Event-tracking schemes

The case of a single contact.

# The case of a single contact.

### Comments

The Delassus example.

In the one-contact case, a naive approach consists in to suppressing the constraint if  $F_{k+1} < 0$  after a integration with a bilateral constraints.

→ Work only for the one contact case.

► The role of the "ε"

In practical situation, all of the test are made up to an accuracy threshold. All statements of the type g = 0 are replaced by  $|g| < \varepsilon$ . The role of these epsilons can be very important and they are quite difficult to size.

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The case of a single contact.

# The case of a single contact.

## Comments

• If the ODE solvers is able to perform the root finding of the function g = 0 for  $status_k = 0$  and  $F^+ = 0$  for  $status_k = 1$ 

→ the case 1.4, 1.5 and the case 2.2.2 can be suppressed.

- If the drift from the constraints is also controlled into the ODE solver by a error computation,
  - → the case 2.1 can also be suppressed
- Most of the case can be resumed into the following step
  - Continue with the same status
  - Compute  $U_{N,k+1}$ ,  $P_{k+1}$  thanks to the LCP (9)(impact equations).
  - Compute  $\Gamma_{N,k+1}$ ,  $F_{k+1}$  thanks to the LCP (20) (Smooth dynamics)
- → Rearranging the cases, we obtain the following algorithm.

Event-tracking schemes

The case of a single contact.

# The case of a single contact. An algorithm

**Require:**  $(g_k, U_N, status_k)$ **Ensure:**  $(g_{k+1}, U_{N,k+1}, status_{k+1})$ Time-integration of the system on  $[t_k, t_{k+1}](22)$  if  $status_k = 0$  or of the system (23) if  $status_k = 1$  up to an event. if  $g_{k+1} > 0$  then  $status_{k+1} = 0$  //The constraint is still not active. (case 1.1) end if if  $g_{k+1} = 0$ ,  $U_{N,k+1} < 0$  then //The constraint is active  $g_{k+1} = 0$  and an impact occur  $U_{N,k+1} < 0$  (case 1.2) Solve the LCP (9) for  $U_{N}^{-} = U_{N,k+1}$ ;  $U_{N,k+1} = U_{N}^{+}$ if  $U_{N,k+1} > 0$  then  $status_{k+1} = 0$ end if if  $g_{k+1} = 0$ ,  $U_{N,k+1} = 0$  then //The constraint is active  $g_{k+1} = 0$  without impact (case 1.2.2, case 1.3, case 2.2) solve the LCP (21) if  $\Gamma_{N,k+1} = 0, F_{k+1} > 0$  then  $status_{k+1} = 1$ else if  $\Gamma_{N,k+1} > 0, F_{k+1} = 0$  then  $status_{k+1} = 0$ else if  $\Gamma_{N,k+1} = 0, F_{k+1} = 0$  then //Undetermined case. end if イロト イポト イヨト イヨト ヨー ク end if Numerical methods for nonsmooth mechanical systems Vincent Acary, INRIA Rhône-Alpes, Grenoble. - 20/127

L The multi-contact case and the index-sets

# The multi-contact case and the index-sets

#### Index sets

The index set I is the set of all unilateral constraints in the system

$$I = \{1 \dots \nu\} \subset \mathbb{N} \tag{24}$$

The index-set  $I_c$  is the set of all active constraints of the system,

$$I_c = \{ \alpha \in I, g^{\alpha} = 0 \} \subset I$$
(25)

and the index-set  $I_s$  is the set of all active constraints of the system with a relative velocity equal to zero,

$$I_{s} = \{ \alpha \in I_{c}, U_{\mathsf{N}}^{\alpha} = \mathsf{0} \} \subset I_{c}$$

$$\tag{26}$$

Event-tracking schemes

The multi-contact case and the index-sets

### The multi-contact case and the index-sets

#### Impact equations

$$\begin{cases} \mathcal{M}(q(t_i))(v^+(t_i) - v^-(t_i)) = p_i, \\ U_{\mathsf{N}}^+(t_i) = \nabla_q g^T(q(t_i))v^+(t_i) \\ U_{\mathsf{N}}^-(t_i) = \nabla_q g^T(q(t_i))v^-(t_i) \\ p_i = \nabla_q g(q(t_i))P_{\mathsf{N},i} \\ \mathcal{P}_{\mathsf{N},i}^\alpha = 0; U_{\mathsf{N}}^{\alpha,+}(t_i) = U_{\mathsf{N}}^{\alpha,-}(t_i), \quad \forall \alpha \in I \setminus I_c \\ 0 \leqslant U_{\mathsf{N}}^{+,\alpha}(t_i) + eU_{\mathsf{N}}^{-,\alpha}(t_i) \perp P_{\mathsf{N},i}^\alpha \ge 0, \quad \forall \alpha \in I_c \end{cases}$$

$$(27)$$

Using the fact that  $P_{N,i}^{\alpha} = 0$  for  $\alpha \in I \setminus I_c$ , this problem can be reduced on the local unknowns  $U_N^+(t_i), P_{N,i} \quad \forall \alpha \in I_c$ .

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Event-tracking schemes

L The multi-contact case and the index-sets

# The multi-contact case and the index-sets Modes for the smooth Dynamics

The smooth unilateral dynamics as a LCP

$$\begin{cases}
M(q)\gamma^{+} + F_{int}(\cdot, q, v) = F_{ext} + \nabla_{q}g(q)F^{+} \\
\Gamma_{N}^{+} = \nabla_{q}g^{T}(q)\gamma^{+} + \frac{d}{dt}(\nabla_{q}g^{T}(q))v^{+} \\
F^{+,\alpha} = 0, \quad \forall \alpha \in I \setminus I_{s} \\
0 \leqslant \Gamma_{N}^{+,\alpha} \perp F^{+,\alpha} \ge 0 \quad \forall \alpha \in I_{s}
\end{cases}$$
(28)

The smooth bilateral dynamics

$$\begin{cases}
M(q)\gamma^{+} + F_{int}(\cdot, q, v) = F_{ext} + \nabla_{q}g(q)F^{+} \\
\Gamma_{N}^{+} = \nabla_{q}g^{T}(q)\gamma^{+} + \frac{d}{dt}(\nabla_{q}g^{T}(q))v^{+} \\
F^{+,\alpha} = 0, \quad \forall \alpha \in I \setminus I_{s}
\end{cases}$$
(29)

Numerical methods for nonsmooth mechanical states  $\overline{\nabla n}$   $\overline{\nabla n$ 

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Event-tracking schemes

- The multi-contact case and the index-sets

## The multi-contact case and the index-sets. an algorithm

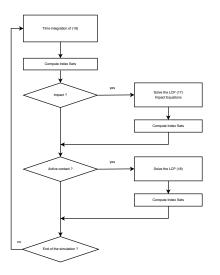
```
Require: (g_k, U_{N,k}, I_{c,k}, I_{s,k}),
Ensure: (g_{k+1}, U_{N,k+1}, I_{c,k+1}, I_{s,k+1})
   Time-integration on [t_k, t_{k+1}] of the system (29) according to I_{c,k} and I_{s,k} up to an
  event.
  Compute the temporary index-sets I_{c,k+1} and I_{s,k+1}.
  if I_{c,k+1} \setminus I_{s,k+1} \neq \emptyset then
     //Impacts occur.
     Solve the LCP (27).
      Update the index-set I_{c,k+1} and temporary I_{s,k+1}
      Check that I_{c,k+1} \smallsetminus I_{s,k+1} = \emptyset
  end if
  if I_{s,k+1} \neq \emptyset then
      Solve the LCP (28)
      for \alpha \in I_{s,k+1} do
         if \Gamma_{N,\alpha,k+1} > 0, F_{\alpha,k+1} = 0 then
            remove \alpha from I_{s,k+1} and I_{c,k+1}
         else if \Gamma_{N,\alpha,k+1} = 0, F_{\alpha,k+1} = 0 then
            //Undetermined case.
         end if
     end for
  end if
   // Go to the next time step
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```

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Event-tracking schemes

L The multi-contact case and the index-sets

# The multi-contact case and the index-sets



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Comments and extensions

# Comments and extensions

## Extensions to Coulomb's friction

The set  $I_r$  is the set of sticking or rolling contact:

$$I_{r} = \{ \alpha \in I_{s}, U_{\mathbb{N}}^{\alpha} = 0, \|U_{\mathbb{T}}\| = 0 \} \subset I_{s},$$
(30)

is the set of sticking or rolling contact, and

$$I_t = \{ \alpha \in I_s, U_N^\alpha = 0, \|U_T\| > 0 \} \subset I_s,$$

$$(31)$$

is the set of slipping or sliding contact.

#### Remarks

In the 3D case, checking the events and the transition sticking/sliding and sliding/sticking is not a easy task.

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Event-tracking schemes

Comments and extensions

# Comments

## Advantages and Weaknesses and the Event Driven schemes

- Advantages :
  - Low cost implementation of time integration solvers (re-use of existing ODE solvers).
  - Higher-order accuracy on free motion.
  - Pseudo-localization of the time of events with finite time-step.
- Weaknesses
  - Numerous events in short time.
  - Accumulation of impacts.
  - No convergence proof
  - Robustness with the respect to thresholds "\varepsilon". Tuning codes is difficult.

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Event-tracking schemes

 $\square$  Comments and extensions

# Event-Capturing (Time-stepping) schemes

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Time-stepping schemes

L Time Discretization of the nonsmooth dynamics

## Time Discretization of the nonsmooth dynamics

For sake of simplicity, the linear time invariant case is only considered.

$$\begin{cases} Mdv + (Kq + Cv^+) dt = F_{ext} dt + di.\\ v^+ = \dot{q}^+ \end{cases}$$
(32)

Integrating both sides of this equation over a time step  $]t_k, t_{k+1}]$  of length h,

$$\begin{cases} \int_{]t_{k},t_{k+1}]} Mdv + \int_{t_{k}}^{t_{k+1}} Cv^{+} + Kq \, dt = \int_{t_{k}}^{t_{k+1}} F_{ext} \, dt + \int_{]t_{k},t_{k+1}]} di , \\ q(t_{k+1}) = q(t_{k}) + \int_{t_{k}}^{t_{k+1}} v^{+} \, dt . \end{cases}$$
(33)

By definition of the differential measure dv,

$$\int_{]t_k,t_{k+1}]} M \, dv = M \int_{]t_k,t_{k+1}]} dv = M \left( v^+(t_{k+1}) - v^+(t_k) \right). \tag{34}$$

Note that the right velocities are involved in this formulation.

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- Time-stepping schemes
  - L Time Discretization of the nonsmooth dynamics

## Time Discretization of the nonsmooth dynamics

The equation of the nonsmooth motion can be written under an integral form as:

$$\begin{cases} M\left(v(t_{k+1}) - v(t_{k})\right) = \int_{t_{k}}^{t_{k+1}} -Cv^{+} - Kq + F_{ext} dt + \int_{]t_{k}, t_{k+1}]} di, \\ q(t_{k+1}) = q(t_{k}) + \int_{t_{k}}^{t_{k+1}} v^{+} dt. \end{cases}$$
(35)

The following notations will be used:

- $q_k \approx q(t_k)$  and  $q_{k+1} \approx q(t_{k+1})$ ,
- $v_k \approx v^+(t_k)$  and  $v_{k+1} \approx v^+(t_{k+1})$ ,

#### Impulse as primary unknown

The impulse  $\int_{]t_k, t_{k+1}]} di$  of the reaction on the time interval  $]t_k, t_{k+1}]$  emerges as a natural unknown. we denote

$$p_{k+1} \approx \int_{]t_k, t_{k+1}]} di$$

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Time-stepping schemes

L Time Discretization of the nonsmooth dynamics

## Time Discretization of the nonsmooth dynamics

#### Interpretation

The measure *di* may be decomposed as follows :

$$di = f dt + pd\nu$$

where

- f dt is the abs. continuous part of the measure di, and
- $pd\nu$  the atomic part.

Two particular cases:

• Impact at  $t_* \in ]t_k, t_{k+1}]$ : If f = 0 and  $pd\nu = p\delta_{t_{k+1}}$  then

$$p_{k+1} = p$$

• Continuous force over  $]t_k, t_{k+1}]$ : If di = fdt and p = 0 then

$$p_{k+1} = \int_{t_k}^{t_{k+1}} f(t) dt$$

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Time-stepping schemes

Time Discretization of the nonsmooth dynamics

# Time Discretization of the nonsmooth dynamics

## Remark

 A pointwise evaluation of a (Dirac) measure is a non sense. It practice using the value

$$f_{k+1} \approx f(t_{k+1})$$

yield severe numerical inconsistencies, since

$$\lim_{h\to 0}f_{k+1}=+\infty$$

Since discontinuities of the derivative v are to be expected if some shocks are occurring, i.e. di has some Dirac atoms within the interval ]t<sub>k</sub>, t<sub>k+1</sub>], it is not relevant to use high order approximations integration schemes for di. It may be shown on some examples that, on the contrary, such high order schemes may generate artefact numerical oscillations.

Time-stepping schemes

L Time Discretization of the nonsmooth dynamics

# Time Discretization of the nonsmooth dynamics

## Discretization of smooth terms

 $\theta\text{-method}$  is used for the term supposed to be sufficiently smooth,

$$\int_{t_k}^{t_{k+1}} Cv + Kq \, dt \approx h \left[ \theta (Cv_{k+1} + Kq_{k+1}) + (1-\theta) (Cv_k + Kq_k) \right]$$

$$\int_{t_k}^{t_{k+1}} F_{ext}(t) \, dt \approx h \left[ \theta (F_{ext})_{k+1} + (1-\theta) (F_{ext})_k \right]$$

The displacement, assumed to be absolutely continuous is approximated by:

$$q_{k+1} = q_k + h \left[ \theta v_{k+1} + (1 - \theta) v_k \right]$$
.

Time-stepping schemes

L Time Discretization of the nonsmooth dynamics

## Time Discretization of the nonsmooth dynamics

Finally, introducing the expression of  $q_{k+1}$  in the first equation of (34), one obtains:

$$\begin{bmatrix} M + h\theta C + h^{2}\theta^{2}K \end{bmatrix} (v_{k+1} - v_{k}) = -hCv_{k} - hKq_{k} - h^{2}\theta Kv_{k} + h[\theta(F_{ext})_{k+1}) + (1 - \theta)(F_{ext})_{k}] + p_{k+1},$$
(36)

which can be written :

$$v_{k+1} = v_{free} + \widehat{M}^{-1} p_{k+1}$$
(37)

where,

- ▶ the matrix  $\widehat{M} = [M + h\theta C + h^2 \theta^2 K]$  is usually called the iteration matrix and,
- The vector

$$v_{free} = v_k + \widehat{M}^{-1} \left[ -hCv_k - hKq_k - h^2\theta Kv_k + h\left[\theta(F_{ext})_{k+1}\right] + (1-\theta)(F_{ext})_k \right] \right]$$

is the so-called "free" velocity, i.e. the velocity of the system when reaction forces are null.

Time-stepping schemes

L Time Discretization of the kinematics relations

# Time Discretization of the kinematics relations

According to the implicit mind, the discretization of kinematic laws is proposed as follows.

For a constraint  $\alpha$ ,

$$U_{k+1}^{\alpha} = H^{\alpha T}(q_k) v_{k+1}$$

$$p_{k+1}^{\alpha} = H^{\alpha}(q_k) P_{k+1}^{\alpha}, \quad p_{k+1} = \sum_{\alpha} p_{k+1}^{\alpha},$$

where

$$P_{k+1}^{\alpha} \approx \int_{]t_k, t_{k+1}]} d\lambda^{\alpha}.$$

For the unilateral constraints, it is proposed

$$g_{k+1}^{lpha} = g_k^{lpha} + h \left[ heta U_{k+1}^{lpha} + (1- heta) U_k^{lpha} 
ight] \, .$$

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Time-stepping schemes

L Discretization of the unilateral constraints

## Discretization of the unilateral constraints

Recall that the unilateral constraint is expressed in terms of velocity as

$$-di \in N_{\mathcal{T}_{\mathcal{C}}(q)}(v^{+}) \tag{38}$$

or in local coordinates as

$$-d\lambda^{\alpha} \in N_{\mathcal{T}_{\mathrm{IR}_{+}}(g(q))}(U^{\alpha,+})$$
(39)

The time discretization is performed by

$$-P_{k+1}^{\alpha} \in N_{\mathcal{T}_{\mathrm{IR}^+}(g^{\alpha}(\tilde{q}_{k+1}))}(U_{k+1}^{\alpha})$$

$$\tag{40}$$

where  $\tilde{q}_{k+1}$  is a forecast of the position for the activation of the constraints, for instance,

$$ilde{q}_{k+1} = q_k + rac{h}{2}v_k$$

In the complementarity formalism, we obtain

$$\text{if } g^{\alpha}(\tilde{q}_{k+1}) \leqslant 0, \text{ then } \quad 0 \leqslant U_{k+1}^{\alpha} \perp P_{k+1}^{\alpha} \geqslant 0$$

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# Summary of the time discretized equations

One step linear problem 
$$\begin{cases} v_{k+1} = v_{free} + \widehat{M}^{-1} p_{k+1} \\ q_{k+1} = q_k + h \left[ \theta v_{k+1} + (1 - \theta) v_k \right] \end{cases}$$
Relations 
$$\begin{cases} U_{k+1}^{\alpha} = H^{\alpha T}(q_k) v_{k+1} \\ p_{k+1}^{\alpha} = H^{\alpha}(q_k) P_{k+1}^{\alpha} \end{cases}$$
Nonsmooth Law 
$$\begin{cases} \text{if } g^{\alpha}(\tilde{q}_{k+1}) \leq 0, \text{ then} \\ 0 \leq U_{k+1}^{\alpha} \perp P_{k+1}^{\alpha} \geq 0 \end{cases}$$

One step LCP

$$U_{k+1} = H^{T}(q_k) v_{\text{free}} + H^{T}(q_k) \widehat{M}^{-1} H(q_k) P_{k+1}$$

$$\text{if } g_p^{\alpha} \leqslant 0, \text{ then } 0 \leqslant U_{k+1}^{\alpha} \perp P_{k+1}^{\alpha} \geqslant 0$$

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Time-stepping schemes

Moreau's time-stepping

# Moreau's Time stepping scheme

$$M(q_{k+\theta})(v_{k+1}-v_k)-h\tilde{F}_{k+\theta}=H(q_{k+\theta})P_{k+1}, \tag{41a}$$

$$q_{k+1} = q_k + h v_{k+\theta}, \tag{41b}$$

$$U_{k+1} = H^{T}(q_{k+\theta}) v_{k+1}$$
(41c)

$$-P_{k+1} \in \partial \psi_{\mathcal{T}_{\mathrm{IR}^{m}_{+}}(\tilde{y}_{k+\gamma})}(U_{k+1} + eU_{k}), \tag{41d}$$

$$\tilde{y}_{k+\gamma} = y_k + h\gamma U_k, \quad \gamma \in [0,1].$$
 (41e)

with  $\theta \in [0, 1], \gamma \ge 0$  and  $x_{k+\alpha} = (1 - \alpha)x_{k+1} + \alpha x_k$  and  $\tilde{y}_{k+\gamma}$  is a prediction of the constraints.

#### Properties

- Convergence results for one constraints
- Convergence results for multiple constraints problems with acute kinetic angles
- No theoretical proof of order

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Time-stepping schemes

Schatzman-Paoli's scheme

# Schatzman–Paoli's Time stepping scheme

$$\mathcal{L} M(q_{k}+1)(q_{k+1}-2q_{k}+q_{k-1})-h^{2}F(t_{k+\theta},q_{k+\theta},v_{k+\theta})=p_{k+1}, \quad (42a)$$

$$v_{k+1} = \frac{q_{k+1} - q_{k-1}}{2h},\tag{42b}$$

$$\left(-p_{k+1}\in N_{K}\left(\frac{q_{k+1}+eq_{k-1}}{1+e}\right),$$
(42c)

where  $N_K$  defined the normal cone to K. For  $K = \{q \in \mathbb{R}^n, y = g(q) \ge 0\}$ 

$$0 \leq g\left(\frac{q_{k+1} + eq_{k-1}}{1+e}\right) \perp \nabla g\left(\frac{q_{k+1} + eq_{k-1}}{1+e}\right) P_{k+1} \geq 0$$
(43)

#### Properties

- Convergence results for one constraints
- Convergence results for multiple constraints problems with acute kinetic angles
- No theoretical proof of order

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# Academic examples

## The bouncing Ball and the linear impacting oscillator

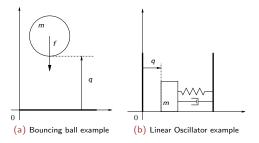


Figure: Academic test examples with analytical solutions

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L Time-stepping schemes

Empirical order

# Academic examples

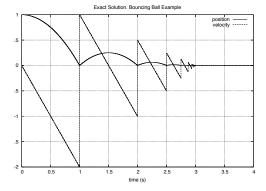


Figure: Analytical solution. Bouncing ball example

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Time-stepping schemes

Empirical order

# Academic examples

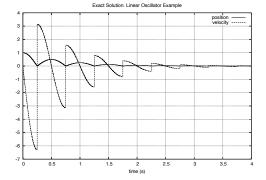


Figure: Analytical solution. Linear Oscillator

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Empirical order

# Measuring error and convergence

Convergence in the sense of filled-in graph (Moreau (1978))

$$gr^{\star}(f) = \{(t,x) \in [0,T] \times \mathbb{R}^n, 0 \leq t \leq T \text{ and } x \in [f(t^-), f(t^+)]\}.$$
(44)

Such graphs are closed bounded subsets of  $[0, T] \times \mathbb{R}^n$ , hence, we can use the Hausdorff distance between two such sets with a suitable metric:

$$d((t,x),(s,y)) = \max\{|t-s|, ||x-y||\}.$$
(45)

Defining the excess of separation between two graphs by

$$e(gr^{\star}(f), gr^{\star}(g)) = \sup_{(t,x) \in gr^{\star}(f)} \inf_{(s,y) \in gr^{\star}(g)} d((t,x), (s,y)),$$
(46)

the Hausdorff distance between two filled-in graphs  $h^{\star}$  is defined by

$$h^{\star}(gr^{\star}(f), gr^{\star}(g)) = \max\{e(gr^{\star}(f), gr^{\star}(g)), e(gr^{\star}(g), gr^{\star}(f))\}.$$
(47)

Fmpirical order

# Measuring error and convergence

An equivalent grid-function norm to the function norm in  $\mathcal{L}_1$ 

$$\|e\|_{1} = h \sum_{i=0}^{N} |f_{i} - f(t_{i})|$$
(48)

In the same way, the p - norm can be defined by

$$\|e\|_{p} = \left(h\sum_{i=0}^{N} |f_{i} - f(t_{i})|^{p}\right)^{1/p}$$
(49)

The computation of this two last norm is easier to implement for piecewise continuous analytical function than the Hausdorff distance.

#### Global order of convergence.

#### Definition

A one-step time-integration scheme is of order q for a given norm  $\|\cdot\|$  if there exists a constant C such that

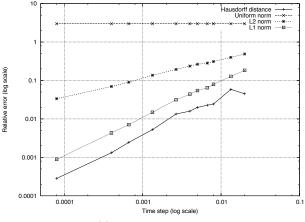
$$\|e\| = Ch^q + \mathcal{O}(h^{q+1}) \tag{50}$$

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Empirical order

# Empirical order of convergence. Moreau's time-stepping scheme



(a) The bouncing ball example

Figure: Empirical order of convergence of the Moreau's time-stepping scheme.

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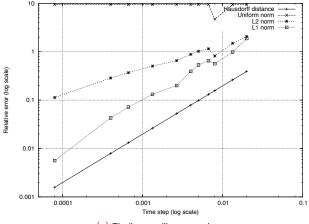
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Empirical order

# Empirical order of convergence. Moreau's time-stepping scheme



(a) The linear oscillator example

Figure: Empirical order of convergence of the Moreau's time-stepping scheme.

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Empirical order

# Empirical order of convergence. Schatzman-Paoli's time-stepping scheme

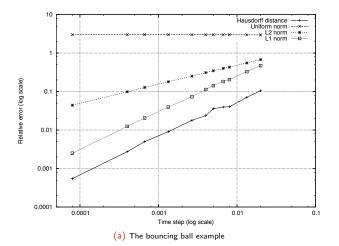


Figure: Empirical order of convergence of the Schatzman-Paoli's time-stepping scheme.

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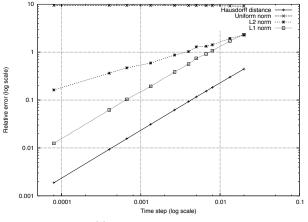
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Time-stepping schemes

Empirical order

# Empirical order of convergence. Schatzman-Paoli's time-stepping scheme



(a) The linear oscillator example



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L Time-stepping schemes

Empirical order

# Comparison

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# State-of-the-art

 $\label{eq:numerical time-integration methods for Nonsmooth Multibody systems (NSMBS):$ 

Nonsmooth event capturing methods (Time-stepping methods)

- $\oplus \,$  robust, stable and proof of convergence
- $\oplus$  low kinematic level for the constraints
- $\oplus\,$  able to deal with finite accumulation
- $\ominus$  very low order of accuracy even in free flight motions

## Nonsmooth event tracking methods (Event-driven methods)

- $\oplus$  high level integration of free flight motions
- $\ominus$  no proof of convergence
- $\ominus$  sensibility to numerical thresholds
- $\ominus$  reformulation of constraints at higher kinematic levels.
- $\ominus\,$  unable to deal with finite accumulation

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# Newmark-type schemes for flexible multibody systems and FEM applications.

Joint work with O. Brüls, Q.Z. Chen and G. Virlez (Université de Liège)

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Newmark-type schemes for flexible multibody systems

└─ Newmark's scheme.

# The Newmark scheme

# Linear Time "Invariant" Dynamics without contact

$$\begin{cases} M\dot{v}(t) + Kq(t) + Cv(t) = f(t) \\ \dot{q}(t) = v(t) \end{cases}$$
(51)

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Newmark-type schemes for flexible multibody systems

Newmark's scheme.

# The Newmark scheme (Newmark, 1959)

#### Principle

Given two parameters  $\gamma$  and  $\beta$ 

$$\begin{cases}
\mathsf{Ma}_{k+1} = f_{k+1} - \mathsf{K}q_{k+1} - \mathsf{C}v_{k+1} \\
\mathsf{v}_{k+1} = \mathsf{v}_k + h\mathsf{a}_{k+\gamma} \\
\mathsf{q}_{k+1} = \mathsf{q}_k + h\mathsf{v}_k + \frac{h^2}{2}\mathsf{a}_{k+2\beta}
\end{cases}$$
(52)

# Notations

$$f(t_{k+1}) = f_{k+1}, \quad x_{k+1} \approx x(t_{k+1}), x_{k+\gamma} = (1-\gamma)x_k + \gamma x_{k+1}$$
(53)

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Newmark-type schemes for flexible multibody systems

Newmark's scheme.

# The Newmark scheme

#### Implementation

Let us consider the following explicit prediction

$$\begin{cases} v_k^* = v_k + h(1 - \gamma)a_k \\ q_k^* = q_k + hv_k + \frac{1}{2}(1 - 2\beta)h^2a_k \end{cases}$$
(54)

The Newmark scheme may be written as

$$\begin{cases} a_{k+1} = \hat{M}^{-1}(-Kq_k^* - Cv_k^* + f_{k+1}) \\ v_{k+1} = v_k^* + h\gamma a_{k+1} \\ q_{k+1} = q_k^* + h^2\beta a_{k+1} \end{cases}$$
(55)

with the iteration matrix

$$\hat{M} = M + h^2 \beta K + \gamma h C \tag{56}$$

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Newmark-type schemes for flexible multibody systems

Newmark's scheme.

# The Newmark scheme

# Properties

- One-step method in state. (Two steps in position)
- Second order accuracy if and only if  $\gamma = \frac{1}{2}$
- Unconditional stability for  $2\beta \ge \gamma \ge \frac{1}{2}$

Average acceleration (Trapezoidal rule)	implicit	$\gamma = rac{1}{2}$ and $\beta = rac{1}{4}$
central difference	explicit	$\gamma=rac{1}{2}$ and $eta=0$
linear acceleration	implicit	$\gamma=\frac{1}{2} \text{ and } \beta=\frac{1}{6}$
Fox–Goodwin (Royal Road)	implicit	$\gamma = rac{1}{2}$ and $\beta = rac{1}{12}$

Table: Standard value for Newmark scheme ((Hughes, 1987, p 493)Géradin and Rixen (1993))

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Newmark-type schemes for flexible multibody systems

Newmark's scheme.

# The Newmark scheme

## High frequencies dissipation

- In flexible multibody Dynamics or in standard structural dynamics discretized by FEM, high frequency oscillations are artifacts of the semi-discrete structures.
- ▶ In Newmark's scheme, maximum high frequency damping is obtained with

$$\gamma \gg \frac{1}{2}, \quad \beta = \frac{1}{4}(\gamma + \frac{1}{2})^2 \tag{57}$$

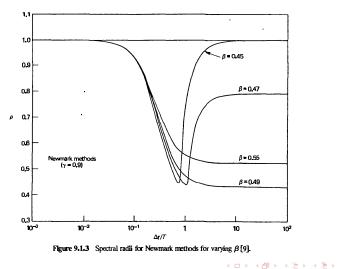
example for  $\gamma = 0.9$ ,  $\beta = 0.49$ 

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Newmark's scheme.

# The Newmark scheme

From (Hughes, 1987) :



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Numerical methods for nonsmooth mechanical systems
Newmark-type schemes for flexible multibody systems
HHT scheme

# The Hilber–Hughes–Taylor scheme. Hilber et al. (1977) Objectives

to introduce numerical damping without dropping the order to one.

#### Principle

Given three parameters  $\gamma,\,\beta$  and  $\alpha$  and the notation

$$M\ddot{q}_{k+1} = -(Kq_{k+1} + Cv_{k+1}) + F_{k+1}$$
(58)

$$\begin{cases}
Ma_{k+1} = M\ddot{q}_{k+1+\alpha} = -(Kq_{k+1+\alpha} + Cv_{k+1+\alpha}) + F_{k+1+\alpha} \\
v_{k+1} = v_k + ha_{k+\gamma} \\
q_{k+1} = q_k + hv_k + \frac{h^2}{2}a_{k+2\beta}
\end{cases}$$
(59)

Standard parameters (Hughes, 1987, p532) are

$$\alpha \in [-1/3, 0], \gamma = (1 - 2\alpha/2) \text{ and } \beta = (1 - \alpha)^2/4$$
 (60)

#### Warning

The notation are abusive.  $a_{k+1}$  is not the approximation of the acceleration at  $t_{k+1} = 0.0$ Numerical methods for nonsmooth mechanical systems Vincent Acary, INRIA Rhône-Alpes, Grenoble.

Newmark-type schemes for flexible multibody systems

HHT scheme

# The HHT scheme

## Properties

- Two-step method in state. (Three-steps method in position)
- ▶ Unconditional stability and second order accuracy with the previous rule. (60)
- For  $\alpha = 0$ , we get the trapezoidal rule and the numerical dissipation increases with  $|\alpha|$ .

Newmark-type schemes for flexible multibody systems

HHT scheme

# The HHT scheme

From (Hughes, 1987) :

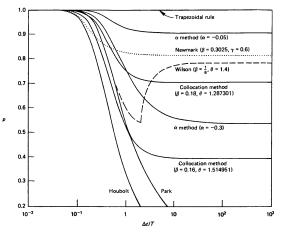


Figure 9.3.1 Spectral radii for  $\alpha$ -methods, optimal collocation schemes, and Houbolt, Newmark, Park, and Wilson methods [22].

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Newmark-type schemes for flexible multibody systems

 $\Box$  Generalized  $\alpha$ -methods

# Generalized $\alpha$ -methods (Chung and Hulbert, 1993) Principle

Given three parameters  $\gamma,~\beta,~\alpha_{\textit{m}}$  and  $\alpha_{\textit{f}}$  and the notation

$$M\ddot{q}_{k+1} = -(Kq_{k+1} + Cv_{k+1}) + F_{k+1}$$
(61)

$$\begin{cases}
Ma_{k+1-\alpha_{m}} = M\ddot{q}_{k+1-\alpha_{f}} \\
v_{k+1} = v_{k} + ha_{k+\gamma} \\
q_{k+1} = q_{k} + hv_{k} + \frac{h^{2}}{2}a_{k+2\beta}
\end{cases}$$
(62)

Standard parameters (Chung and Hulbert, 1993) are chosen as

$$\alpha_m = \frac{2\rho_\infty - 1}{\rho_\infty + 1}, \quad \alpha_f = \frac{\rho_\infty}{\rho_\infty + 1}, \quad \gamma = \frac{1}{2} + \alpha_f - \alpha_m \text{ and } \beta = \frac{1}{4}(\gamma + \frac{1}{2})^2 \quad (63)$$

where  $\rho_\infty \in [0,1]$  is the spectral radius of the algorithm at infinity.

#### Properties

- Two-step method in state.
- Unconditional stability and second order accuracy.
- Optimal combination of accuracy at low-frequency and numerical damping at Numerical methodalighefrequency Anical systems Vincent Acary, INRIA Rhône-Alpes, Grenoble. - 57/127

Newmark-type schemes for flexible multibody systems

 $\Box$  Generalized  $\alpha$ -methods

## A first naive approach

Direct Application of the HHT scheme to Linear Time "Invariant" Dynamics with contact

$$\begin{cases} M\dot{v}(t) + Kq(t) + Cv(t) = f(t) + r(t), \text{ a.e} \\ \dot{q}(t) = v(t) \\ r(t) = G(q) \lambda(t) \\ g(t) = g(q(t)), \quad \dot{g}(t) = G^{T}(q(t))v(t), \\ 0 \leq g(t) \perp \lambda(t) \geq 0, \end{cases}$$
(64)

results in

$$\begin{cases} M\ddot{q}_{k+1} = -(Kq_{k+1} + Cv_{k+1}) + F_{k+1} + r_{k+1} \\ r_{k+1} = G_{k+1}\lambda_{k+1} \end{cases}$$
(65)

$$\begin{cases} Ma_{k+1} = M\ddot{g}_{k+1+\alpha} \\ v_{k+1} = v_k + ha_{k+\gamma} \\ q_{k+1} = q_k + hv_k + \frac{h^2}{2}a_{k+2\beta} \\ 0 \leqslant g_{k+1} \perp \lambda_{k+1} \ge 0, \end{cases}$$
(66)

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# A first naive approach

## Direct Application of the HHT scheme to Linear Time "Invariant" Dynamics with contact

The scheme is not consistent for mainly two reasons:

- If an impact occur between rigid bodies, or if a restitution law is needed which is mandatory between semidiscrete structure, the impact law is not taken into account by the discrete constraint at position level
- Even if the constraint is discretized at the velocity level, i.e.

$$\text{if } \bar{g}_{k+1}, \text{ then } 0 \leqslant \dot{g}_{k+1} + eg_k \perp \lambda_{k+1} \geqslant 0 \tag{67}$$

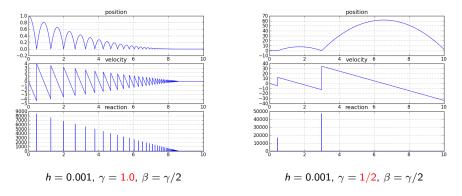
the scheme is consistent only for  $\gamma = 1$  and  $\alpha = 0$  (first order approximation.)

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 $\Box$  Generalized  $\alpha$ -methods

# A first naive approach

Velocity based constraints with standard Newmark scheme ( $\alpha = 0.0$ ) Bouncing ball example. m = 1, g = 9.81,  $x_0 = 1.0 v_0 = 0.0$ , e = 0.9

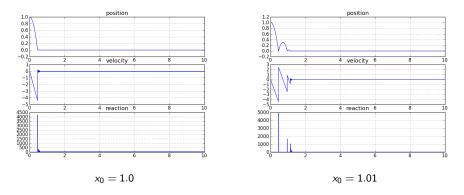


Newmark-type schemes for flexible multibody systems

 $\Box$  Generalized  $\alpha$ -methods

# A first naive approach

Position based constraints with standard Newmark scheme ( $\alpha = 0.0$ ) Bouncing ball example. m = 1, g = 9.81,  $v_0 = 0.0$ , e = 0.9, h = 0.001,  $\gamma = 1.0$ ,  $\beta = \gamma/2$ 



Newmark-type schemes for flexible multibody systems

 $\Box$  Generalized  $\alpha$ -methods

# The Nonsmooth Newmark and HHT scheme

## Dynamics with contact and (possibly) impact

$$\begin{cases}
M \, dv = F(t, q, v) \, dt + G(q) \, di \\
\dot{q}(t) = v^{+}(t), \\
g(t) = g(q(t)), \quad \dot{g}(t) = G^{T}(q(t))v(t), \\
\text{if } g(t) \leq 0, \quad 0 \leq g^{+}(t) + e\dot{g}^{-}(t) \perp di \geq 0,
\end{cases}$$
(68)

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Newmark-type schemes for flexible multibody systems

 $\Box$  Generalized  $\alpha$ -methods

# The Nonsmooth Newmark and HHT scheme

#### Splitting the dynamics between smooth and nonsmooth part

$$M \, dv = Ma(t) \, dt + M \, dv^{\rm con} \tag{69}$$

with

$$\begin{cases} Ma \, dt = F(t, q, v) \, dt \\ M \, dv^{\text{con}} = G(q) \, di \end{cases}$$
(70)

Different choices for the discrete approximation of the term Ma dt and  $M dv^{con}$ 

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Newmark-type schemes for flexible multibody systems

 $\Box$  Generalized  $\alpha$ -methods

# The Nonsmooth Newmark and HHT scheme

## Principles

As usual is the Newmark scheme, the smooth part of the dynamics Ma dt = F(t, q, v) dt is collocated, i.e.

$$Ma_{k+1} = F_{k+1} \tag{71}$$

the impulsive part a first order approximation is done over the time-step

$$M\Delta v_{k+1}^{\rm con} = G_{k+1} \Lambda_{k+1} \tag{72}$$

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# The Nonsmooth Newmark and HHT scheme

## Principles

$$\begin{cases}
\mathsf{Ma}_{k+1} = F_{k+1+\alpha} \\
\mathsf{M}\Delta v_{k+1}^{\text{con}} = G_{k+1} \Lambda_{k+1} \\
\mathsf{v}_{k+1} = \mathsf{v}_{k} + h\mathsf{a}_{k+\gamma} + \Delta \mathsf{v}_{k+1}^{\text{con}} \\
\mathsf{q}_{k+1} = \mathsf{q}_{k} + h\mathsf{v}_{k} + \frac{h^{2}}{2}\mathsf{a}_{k+2\beta} + \frac{1}{2}h\Delta \mathsf{v}_{k+1}^{\text{con}}
\end{cases}$$
(73)

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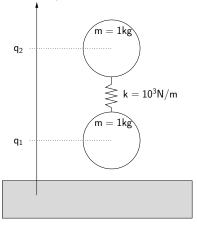
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Newmark-type schemes for flexible multibody systems

 $\Box$  Generalized  $\alpha$ -methods

## The Nonsmooth Newmark and HHT scheme

Example (Two balls oscillator with impact)

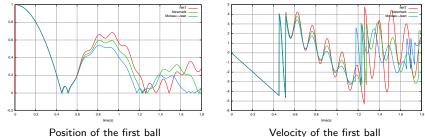


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 $\Box$  Generalized  $\alpha$ -methods

## The Nonsmooth Newmark and HHT scheme

time-step : h = 2e - 3. Moreau ( $\theta = 1.0$ ). Newmark ( $\gamma = 1.0, \beta = 0.5$ ). HHT ( $\alpha = 0.1$ )

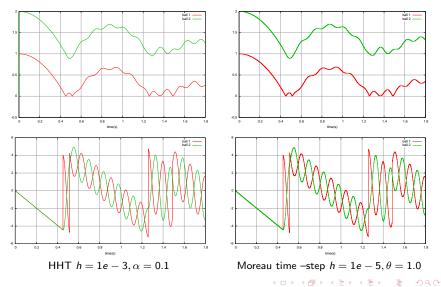


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Newmark-type schemes for flexible multibody systems

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## The Nonsmooth Newmark and HHT scheme



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## The Nonsmooth Newmark and HHT scheme

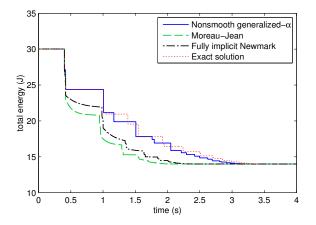


Figure 7. Numerical results for the total energy of the bouncing oscillator.

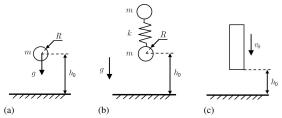
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## The Nonsmooth Newmark and HHT scheme



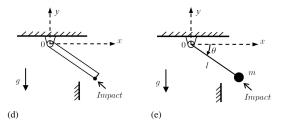


Figure 2. Examples: (a) bouncing ball; (b) linear vertical oscillator; (c) bouncing of an elastic bar; (d) bouncing of a nonlinear beam pendulum; (e)bouncing of a rigid pendulum

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## The Nonsmooth Newmark and HHT scheme

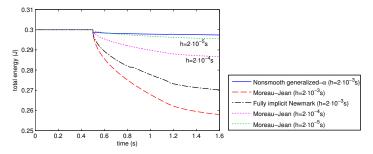


Figure 9. Numerical results for the total energy of the bouncing elastic bar

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 $\Box$  Generalized  $\alpha$ -methods

## The Nonsmooth Newmark and HHT scheme

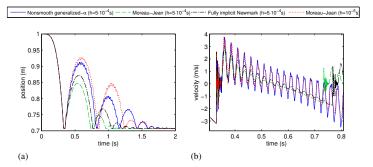


Figure 10. Numerical results for the impact of a flexible rotating beam: (a) position, (b) velocity.

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 $\Box$  Generalized  $\alpha$ -methods

# The Nonsmooth Newmark and HHT scheme

## Observed properties on examples

- the scheme is consistent and globally of order one.
- the scheme seems to share the stability property as the original HHT
- the scheme dissipates energy only in high-frequency oscillations (w.r.t the time-step.)

## Conclusions & perspectives

- Extension to any multi-step schemes can be done in the same way.
- Improvements of the order by splitting.
- Recast into time-discontinuous Galerkin formulation.

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Newmark-type schemes for flexible multibody systems

L-Time-continuous energy balance equations

## Energy analysis

## Time-continuous energy balance equations

Let us start with the "LTI" Dynamics

$$\begin{cases} M \, dv + (Kq + Cv) \, dt = F \, dt + \, di \\ dq = v^{\pm} \, dt \end{cases}$$
(74)

we get for the Energy Balance

$$d(v^{\top}Mv) + (v^{+} + v^{-})(Kq + Cv) dt = (v^{+} + v^{-})F dt + (v^{+} + v^{-}) di$$
(75)

that is

$$2d\mathcal{E} := d(v^{\top}Mv) + 2q^{\top}Kdq = 2v^{\top}F dt - 2v^{\top}Cv dt + (v^{+} + v^{-})^{\top} di$$
(76)

with

$$\mathcal{E} := \frac{1}{2} \mathbf{v}^\top M \mathbf{v} + \frac{1}{2} \mathbf{q}^\top \mathbf{K} \mathbf{q}.$$
(77)

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L-Time-continuous energy balance equations

## Energy analysis

#### Time-continuous energy balance equations

If we split the differential measure in  $di = \lambda \, dt + \sum_i p_i \delta_{t_i}$ , we get

$$2d\mathcal{E} = 2\mathbf{v}^{\top}(\mathbf{F}+\lambda) \ dt - 2\mathbf{v}^{\top}\mathbf{C}\mathbf{v} \ dt + (\mathbf{v}^{+}+\mathbf{v}^{-})^{\top}\mathbf{p}_{i}\delta_{t_{i}}$$
(78)

By integration over a time interval  $[t_0, t_0]$  such that  $t_i \in [t_0, t_1]$ , we obtain an energy balance equation as

$$\Delta \mathcal{E} := \mathcal{E}(t_1) - \mathcal{E}(t_0)$$

$$= \underbrace{\int_{t_0}^{t_1} v^\top F \, dt}_{W^{\text{ext}}} - \underbrace{\int_{t_0}^{t_1} v^\top C v \, dt}_{W^{\text{damping}}} + \underbrace{\int_{t_0}^{t_1} v^\top \lambda \, dt}_{W^{\text{con}}} + \underbrace{\frac{1}{2} \sum_{i} (v^+(t_i) + v^-(t_i))^\top p_i}_{W^{\text{impact}}}$$
(79)

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Newmark-type schemes for flexible multibody systems

- Time-continuous energy balance equations

## Energy analysis

## Work performed by the reaction impulse di

The term

$$W^{\rm con} = \int_{t_0}^{t_1} v^\top \lambda \ dt \tag{80}$$

is the work done by the contact forces within the time-step. If we consider perfect unilateral constraints, we have  $W^{con} = 0$ .

The term

$$W^{\text{impact}} = \frac{1}{2} \sum_{i} (v^{+}(t_{i}) + v^{-}(t_{i}))^{\top} p_{i}$$
(81)

represents the work done by the contact impulse  $p_i$  at the time of impact  $t_i$ . Since  $p_i = G(t_i)P_i$  and if we consider the Newton impact law, we have

$$W^{\text{impact}} = \frac{1}{2} \sum_{i} (v^{+}(t_{i}) + v^{-}(t_{i}))^{\top} G(t_{i}) P_{i}$$
  
$$= \frac{1}{2} \sum_{i} (U^{+}(t_{i}) + U^{-}(t_{i}))^{\top} P_{i}$$
  
$$= \frac{1}{2} \sum_{i} ((1 - e)U^{-}(t_{i}))^{\top} P_{i} \leq 0 \text{ for } 0 \leq e \leq 1$$
  
(82)

with the local relative velocity defines as  $U(t) = G^{\top}(t)v(t) \Rightarrow (z > 4z)$ Numerical methods for nonsmooth mechanical systems Vincent Acary, INRIA Rhône-Alpes, Grenoble. - 76/127

Energy analysis for Moreau-Jean scheme

## Energy analysis for Moreau-Jean scheme

#### Lemma

Let us assume that the dynamics is a LTI dynamics with C = 0. Let us define the discrete approximation of the work done by the external forces within the step (supply rate) by

$$\bar{W}_{k+1}^{\text{ext}} = h \mathbf{v}_{k+\theta}^{\top} F_{k+\theta} \approx \int_{t_k}^{t_{k+1}} F \mathbf{v} \, dt \tag{83}$$

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Then the variation of energy over a time-step performed by the Moreau-Jean is

$$\Delta \mathcal{E} - \bar{W}_{k+1}^{\text{ext}} = \left(\frac{1}{2} - \theta\right) \left[ \|v_{k+1} - v_k\|_M^2 + \|(q_{k+1} - q_k)\|_K^2 \right] + U_{k+\theta}^\top P_{k+1}$$
(84)

Newmark-type schemes for flexible multibody systems

Energy analysis for Moreau-Jean scheme

## Energy analysis for Moreau-Jean scheme

#### Proposition

Let us assume that the dynamics is a LTI dynamics. The Moreau–Jean scheme dissipates energy in the sense that

$$\mathcal{E}(t_{k+1}) - \mathcal{E}(t_k) - \bar{W}_{k+1}^{\text{ext}} \leqslant 0$$
(85)

if

$$\frac{1}{2} \leqslant \theta \leqslant \frac{1}{1+e} \leqslant 1 \tag{86}$$

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In particular, for 
$$e=0,$$
 we get  $rac{1}{2}\leqslant\theta\leqslant 1$  and for  $e=1,$  we get  $heta=rac{1}{2}$  .

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Energy analysis for Moreau-Jean scheme

## Energy analysis for Moreau-Jean scheme

#### Variant of the Moreau scheme that always dissipates energy Let us consider the variant of the Moreau scheme

$$M(v_{k+1} - v_k) + hKq_{k+\theta} - hF_{k+\theta} = p_{k+1} = GP_{k+1},$$
(87a)

$$q_{k+1} = q_k + hv_{k+1/2},$$
 (87b)

$$U_{k+1} = G^{\top} v_{k+1}$$
 (87c)

$$\begin{array}{ll} \text{if} \quad \bar{g}_{k+1}^{\alpha} \leqslant 0 \text{ then } 0 \leqslant U_{k+1}^{\alpha} + eU_{k}^{\alpha} \perp P_{k+1}^{\alpha} \geqslant 0, \\ \text{otherwise } P_{k+1}^{\alpha} = 0. \end{array}$$

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Energy analysis for Moreau-Jean scheme

## Energy analysis for Moreau-Jean scheme

#### Lemma

Let us assume that the dynamics is a LTI dynamics with C = 0. Then the variation of energy performed by the variant scheme over a time-step is

$$\Delta \mathcal{E} - \bar{W}_{k+1}^{\text{ext}} = (\frac{1}{2} - \theta) \| (q_{k+1} - q_k) \|_{K}^2 + U_{k+1/2}^{\top} P_{k+1}$$
(88)

The scheme dissipates energy in the sense that

$$\mathcal{E}(t_{k+1}) - \mathcal{E}(t_k) - \bar{W}_{k+1}^{\text{ext}} \leqslant 0$$
(89)

if

$$\theta \geqslant \frac{1}{2}$$
 (90)

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Energy Analysis for the Newmark scheme

## Energy analysis for Newmark's scheme

#### Lemma

Let us assume that the dynamics is a LTI dynamics with C = 0. Let us define the discrete approximation of the work done by the external forces within the step by

$$\bar{W}_{k+1}^{\text{ext}} = (q_{k+1} - q_k)^\top F_{k+\gamma} \approx \int_{t_k}^{t_{k+1}} F_V \ dt \tag{91}$$

Then the variation of energy over a time-step performed by the scheme is

$$\Delta \mathcal{E} - \bar{W}_{k+1}^{\text{ext}} = (\frac{1}{2} - \gamma) \| (q_{k+1} - q_k) \|_{K}^{2} + \frac{h}{2} (2\beta - \gamma) [(q_{k+1} - q_k)^{\top} K(v_{k+1} - v_k) - (v_{k+1} - v_k)^{\top} [F_{k+1} - F_{k}]] + \frac{1}{2} P_{k+1}^{\top} (U_{k+1} + U_k) + \frac{h}{2} (2\beta - \gamma) (a_{k+1} - a_k)^{\top} GP_{k+1}$$
(92)

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Newmark-type schemes for flexible multibody systems

Energy Analysis for the Newmark scheme

## Energy analysis for Newmark's scheme

Define an discrete "algorithmic energy" (discrete storage function) of the form

$$\mathcal{K}(q, v, a) = \mathcal{E}(q, v) + \frac{h^2}{4} (2\beta - \gamma) a^{\top} M a.$$
(93)

The following result can be given

#### Proposition

Let us assume that the dynamics is a LTI dynamics with C = 0. Let us define the discrete approximation of the work done by the external forces within the step by

$$\bar{W}_{k+1}^{\text{ext}} = (q_{k+1} - q_k)^\top F_{k+\gamma} \approx \int_{t_k}^{t_{k+1}} F_V \ dt \tag{94}$$

Then the variation of energy over a time-step performed by the nonsmooth Newmark scheme is

$$\Delta \mathcal{K} - \bar{W}_{k+1}^{\text{ext}} = -(\gamma - \frac{1}{2}) \left[ \|q_{k+1} - q_k\|_K^2 + \frac{h}{2} (2\beta - \gamma) \|(a_{k+1} - a_k)\|_M^2 \right] + U_{k+1/2}^\top P_{k+1}$$
(95)

Moreover, the nonsmooth Newmark scheme is stable in the following sense

$$\Delta \mathcal{K} - \bar{W}_{k+1}^{\text{ext}} \leqslant 0 \tag{96}$$

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Newmark-type schemes for flexible multibody systems

Energy Analysis for the Newmark scheme

## Energy analysis for HHT scheme

#### Augmented dynamics

Let us introduce the modified dynamics

$$Ma(t) + Cv(t) + Kq(t) = F(t) + \frac{\alpha}{\nu} [Kw(t) + Cx(t) - y(t)]$$
(98)

and the following auxiliary dynamics that filter the previous one

$$\nu h \dot{w}(t) + w(t) = \nu h \dot{q}(t)$$
  

$$\nu h \dot{x}(t) + x(t) = \nu h \dot{v}(t) \qquad (99)$$
  

$$\nu h \dot{y}(t) + y(t) = \nu h \dot{F}(t)$$

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## Energy analysis for HHT scheme

#### Discretized Augmented dynamics

The equation (99) are discretized as follows

$$\nu(w_{k+1} - w_k) + \frac{1}{2}(w_{k+1} + w_k) = \nu(q_{k+1} - q_k)$$
  

$$\nu(x_{k+1} - x_k) + \frac{1}{2}(x_{k+1} + x_k) = \nu(v_{k+1} - v_k)$$
  

$$\nu(y_{k+1} - y_k) + \frac{1}{2}(y_{k+1} + y_k) = \nu(F_{k+1} - F_k)$$
(100)

or rearranging the terms

$$\begin{pmatrix} \frac{1}{2} + \nu \end{pmatrix} w_{k+1} + (\frac{1}{2} - \nu) w_k &= \nu (q_{k+1} - q_k) \\ (\frac{1}{2} + \nu) x_{k+1} + (\frac{1}{2} - \nu) x_k &= \nu (v_{k+1} - v_k) \\ (\frac{1}{2} + \nu) y_{k+1} + (\frac{1}{2} - \nu) y_k &= \nu (F_{k+1} - F_k)$$
 (101)

With the special choice  $\nu = \frac{1}{2}$ , we obtain the HHT scheme collocation that is

$$Ma_{k+1} + (1-\alpha)[Kq_{k+1} + Cv_{k+1}] + \alpha[Kq_k + Cv_k] = (1-\alpha)F_{k+1} + \alpha F_k$$
(102)  
$$= 0 + \langle \mathcal{O} \rangle + \langle \mathcal{$$

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Energy Analysis for the Newmark scheme

## Energy analysis for HHT scheme

# Discretized storage function With

$$\mathcal{H}(q, \mathbf{v}, \mathbf{a}, \mathbf{w}) = \mathcal{E}(q, \mathbf{v}) + \frac{h^2}{4} (2\beta - \gamma) \mathbf{a}^\top M \mathbf{a} + 2\alpha (1 - \gamma) \mathbf{w}^\top K \mathbf{w}.$$
(103)

we get

$$\begin{aligned} 2\Delta \mathcal{H} &= 2U_{k+1/2}^{\top} P_{k+1} \\ &- h^2 (\gamma - \frac{1}{2})(2\beta - \gamma) \| (a_{k+1} - a_k) \|_M^2 \\ &- 2(\gamma - \frac{1}{2} - \alpha) \| q_{k+1} - q_k \|_K^2 \\ &- 2\alpha (1 - 2(\gamma - \frac{1}{2})) \| w_{k+1} - w_k \|_K^2 \\ &+ 2(F_{k+\gamma - \alpha})^{\top} (q_{k+1} - q_k) + 2\alpha (1 - 2(\gamma - \frac{1}{2})) (q_{k+1} - q_k)^{\top} (y_{k+1} - y_k) \end{aligned}$$

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Newmark-type schemes for flexible multibody systems

Energy Analysis for the Newmark scheme

## Energy analysis for HHT scheme

# Discretized storage function With

$$\mathcal{H}(q, v, a, w) = \mathcal{E}(q, v) + \frac{h^2}{4} (2\beta - \gamma) a^{\top} M a + 2\alpha (1 - \gamma) w^{\top} K w.$$
(103)

and with  $\alpha = \gamma - \frac{1}{2}$ , we obtain

$$2\Delta \mathcal{H} = 2U_{k+1/2}^{\top} P_{k+1} - h^{2}(\alpha)(2\beta - \gamma) ||(a_{k+1} - a_{k})||_{M}^{2} - 2\alpha(1 - 2\alpha) ||w_{k+1} - w_{k}||_{K}^{2} + 2(F_{k+\gamma-\alpha})^{\top}(q_{k+1} - q_{k}) + 2\alpha(1 - 2\alpha)(q_{k+1} - q_{k})^{\top}(y_{k+1} - y_{k})$$
(104)

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Newmark-type schemes for flexible multibody systems

Energy Analysis for the Newmark scheme

# Energy analysis for HHT scheme

## Conclusions

- For the Moreau–Jean, a simple variant allows us to obtain a scheme which always dissipates energy.
- ▶ For the Newmark and the HHT scheme with retrieve the dissipation properties as the smooth case. The term associated with impact is added is the balance.
- Open Problem: We get dissipation inequality for discrete with quadratic storage function and plausible supply rate. The nest step is to conclude to the stability of the scheme with this argument.

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Energy Analysis for the Newmark scheme

## Adaptive time-step strategies for time-stepping schemes

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Adaptive schemes

Smooth ODE time integration

# Smooth ODEs

## One-step numerical solvers for ODEs

Let us consider a ODE

$$\dot{x} = f(x, t), \tag{105}$$

where f is a mapping with sufficient regularity. The one-step time-stepping method over the time-step  $[t_k, t_{k+1} = t_k + h]$  is generically denoted by

$$x_{k+1} = x_k + h\Phi(t_k, h, x_k).$$
(106)

## Order of consistency

The one-step time-stepping method is said to be consistent if  $\Phi(t, 0, x, x) = f(x, t)$ and has a consistency order p if there exists a constant C such that

$$e_{k+1} = x(t_{k+1}) - x_{k+1} = Ch^{p+1} + \mathcal{O}(h^{p+2}),$$
(107)

assuming that  $x_k = x(t_k)$ .

If the time-stepping method has an order of consistency p and converges, then the global order of convergence is p,

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Adaptive schemes

Smooth ODE time integration

# Smooth ODEs

## Basic practical error evaluation

- 1. Two "small" time steps of size  $h/2 \implies x_{1/2}$ .
- 2. One "big" time-step  $h \implies x_1$ .

$$e_{1} = x(t_{0} + h) - x_{1} = C h^{p+1} + \mathcal{O}(h^{p+2}),$$
  

$$e_{1/2} = x(t_{0} + h) - x_{1/2} = 2C (h/2)^{p+1} + \mathcal{O}(h^{p+2}).$$
(108)

This procedure permits us to evaluate the constant C and to obtain and a local error estimate such that:

$$e_2 = x(t_0 + h) - x_2 = \frac{x_{1/2} - x_1}{2^p - 1} + \mathcal{O}(h^{p+2}).$$
(109)

## Enhanced practical error evaluation

- Runge–Kutta Embedded pairs (Dormand-Price, Felhberg)
- Milne's device
- Nordsieck's method

Numerical methods for nonsmooth mechanical systems
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# Smooth ODEs

## Automatic control of the time-step

$$\|e_k\| \leqslant etol = atol + rtol \circ \max(x_0, x_k)$$
(110)

The measure of the error is given by

$$error = \|e_k \circ invtol\| \tag{111}$$

with  $invtol = [1/etol_i, i = 1...n]$ . The optima step size is then obtained by

$$h_{\rm opt} = h(\frac{1}{\rm error})^{1/(p+1)}$$
 (112)

Usually, the step size is not allowed to decrease of to increase too fast, thanks to the following heuristic rule

$$h_{\text{new}} = h \min(\alpha_{max}, \max(\alpha_{min}, \alpha(\frac{1}{\text{error}})^{1/(p+1)}))$$
(113)

where  $\alpha, \alpha_{min}$  and  $\alpha_{max}$  are some user parameters.

Adaptive schemes

Local error estimates for the Moreau's Time-stepping scheme

## Local error estimates for the Moreau's time-stepping

## Notation

$$e = x(t_k + h) - x_{k+1} = \begin{bmatrix} e_v \\ e_q \end{bmatrix} = \begin{bmatrix} v^+(t_k + h) - v_{k+1} \\ q(t_k + h) - q_{k+1} \end{bmatrix}$$
(114)

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Local error estimates for the Moreau's Time-stepping scheme

## Local error estimates for the Moreau's time-stepping

## Assumption 1 : Existence and uniqueness

A unique global solution over [0, T] for Moreau's sweeping process is assumed such that  $q(\cdot)$  is absolutely continuous and admits a right velocity  $v^+(\cdot)$  at every instant t of [0, T] and such that the function  $v^+ \in LBV([0, T], \mathbb{R}^n)$ .

→ Assumption 92 is ensured in the framework introduced by Ballard (Ballard, 2000) who proves the existence and uniqueness of a solution in a general framework mainly based on the analyticity of data.

## Assumption 2 : Smoothness of data

The following smoothness on the data will be assumed: a) the inertia operator M(q) is assumed to be of class  $C^p$  and definite positive, b) the force mapping F(t, q, v) is assumed to be of class  $C^p$ , c) the constraint functions g(q) are assumed to be of class  $C^{p+1}$  and d) the Jacobian matrix  $G(q) = \nabla^T_a g(q)$  is assumed to have full-row rank.

Local error estimates for the Moreau's Time-stepping scheme

## Local error estimates for the Moreau's time-stepping

#### Lemma

Let  $I = [t_k, t_{k+1}]$ . Let us assume that the function  $f \in BV(I, \mathbb{R}^n)$ . Then we have the following inequality for the  $\theta$ -method,  $\theta \in [0, 1]$ ,

$$\left\|\int_{t_k}^{t_{k+1}} f(s) \, ds - h(\theta f(t_{k+1}) + (1-\theta)f(t_k))\right\| \leq C(\theta)(t_{k+1} - t_k) \operatorname{var}(f, I), \quad (115)$$

where  $var(f, I) \in \mathbb{R}$  is the variation of f on I and  $C(\theta) = \theta$  if  $\theta \ge 1/2$  and  $C(\theta) = 1 - \theta$  otherwise. Furthermore, the value of  $C(\theta)$  yields a sharp bound in (115).

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Local error estimates for the Moreau's Time-stepping scheme

# Local error estimates for the Moreau's time-stepping

## Proposition

Under Assumptions 1 and 2, the local order of consistency of the Moreau time-stepping scheme for the generalized coordinates is

$$e_q = \mathcal{O}(h)$$

and at least for the velocities

$$e_v = \mathcal{O}(1)$$

## Comments

The bounds are reached if an impact is located within the time-step and the activation of the constraint is not correct.

Adaptive schemes

Local error estimates for the Moreau's Time-stepping scheme

# One impact at time $t_* \in (t_k, t_{k+1}]$ Assumption

$$di = p\delta_{t_*}$$
, or equivalently  $dI = P\delta_{t_*}$ , with  $P = G(t_*)p$ . (116)

Notation

$$\mathcal{I} = \{\alpha, \alpha \in \{1..m\}\}$$
(117)

$$\mathcal{I}_{*} = \{ \alpha \in \mathcal{I}, P^{\alpha} \ge 0, U^{\alpha,+}(t_{*}) - U^{\alpha,-}(t_{*}) = -(1+e)U^{\alpha,-}(t_{*}) \}$$
(118)

$$\mathcal{I}_{p} = \{ \alpha \in \mathcal{I}, P_{k+1}^{\alpha} \ge 0, U_{k+1}^{\alpha} - U_{k}^{\alpha} = -(1+e)U_{k}^{\alpha} \}$$
(119)

#### Lemma

Let us assume that we have only one elastic impact at time  $t_* \in (t_k,t_{k+1}]$  i.e. ,  $di = p\delta_{t_*} + r(t)dt.$ 

1. If  $\mathcal{I}_* = \mathcal{I}_p$ , then the local order of consistency of the scheme is given by

$$e_v = K_v h + \mathcal{O}(h^2) \tag{120}$$

2. If  $\mathcal{I}_* \neq \mathcal{I}_p$ , then the local order of consistency of the scheme is given by

$$e_{v} = K_{v} + \mathcal{O}(h) \quad (121) \quad (121) \quad (121)$$

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Adaptive schemes

Local error estimates for the Moreau's Time-stepping scheme

# Local error estimates for the Moreau's time-stepping Example (The bouncing ball)

$$\begin{cases} \dot{v}(t) = f(t) + \lambda(t), & \dot{q}(t) = v(t), \\ 0 \leqslant q(t) \perp \lambda(t) \ge 0, & v^+(t) = -ev^-(t), \text{ if } q(t) = 0, \end{cases}$$
(122)

With chosen parameters as f = -2, e = 1/2 and the initial data as  $t_0 = 0$ ,  $q_0 = 1$  and  $v_0 = 0$ . The analytical solution reads as

For t ∈ [0, 1),
  

$$\begin{cases}
q(t) = -t^{2} + 1, \\
v(t) = -2t,
\end{cases}$$
(123)
For t ∈  $\left[3 - \frac{1}{2^{n-1}}, 3 - \frac{1}{2^{n}}\right)$ ,
  

$$\begin{cases}
q(t) = -(t-3)^{2} - \frac{3}{2^{n}}(t-1) + \frac{1}{2^{n-1}}\left(3 - \frac{1}{2^{n}}\right), \\
v(t) = -2(t-3) - \frac{3}{2^{n}},
\end{cases}$$
(124)
And for t ∈ [3, +∞)
  

$$\begin{cases}
q(t) = 0, \\
q(t) = 0$$

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Local error estimates for the Moreau's Time-stepping scheme

## Local error estimates for the Moreau's time-stepping

## Example (The bouncing ball (continued))

Let us consider a time interval such that the impacting time  $t_*$  belongs to  $(t_k, t_{k+1}]$ . The error is given by

$$\begin{array}{ll} \text{if} & p_{k+1} = 0 \\ & \left\{ \begin{array}{l} e_v = -(1+e)[v_k + hf\sigma] \\ e_q = -q_k - h(e(1-\sigma+1))v_k - fh^2[e(1-\sigma)\sigma + \frac{1}{2}(1-\sigma)^2 + \theta] \\ \text{if} & p_{k+1} > 0 \end{array} \right. \\ & \left\{ \begin{array}{l} e_v = -hf[1-\sigma - e\sigma] \\ e_q = -q_k - h((1+e)(1-\theta) - e\sigma)v_k - fh^2(e(1-\sigma)\sigma + \frac{1}{2}(1-\sigma)^2) \\ e_q = -q_k - h((1+e)(1-\theta) - e\sigma)v_k - fh^2(e(1-\sigma)\sigma + \frac{1}{2}(1-\sigma)^2) \end{array} \right. \\ & \text{where } \sigma = (t_k - t_k)/h \in (0, 1]. \end{array}$$

The approximate solution of the Moreau scheme depends on the forecast of the active constraints, *i.e.*  $\bar{g}_{k+1} = q_k + \gamma h v_k$ .

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Adaptive schemes

Local error estimates for the Moreau's Time-stepping scheme

## Local error estimates for the Moreau's time-stepping

Example (The bouncing ball (continued)) Using the fact that  $q(t_*) = q_k + v_k \sigma h + \frac{1}{2} (\sigma h)^2 = 0$ , we obtain that  $q_k = -\sigma v_k h - \frac{1}{2} f(\sigma h)^2$  and if  $p_{k+1} = 0.$  $\begin{cases} e_v = -(1+e)[v_k + hf\sigma] \\ e_q = -h(e(1-\sigma+1)-\sigma)v_k - fh^2[e(1-\sigma)\sigma + \frac{1}{2}(1-\sigma)^2 - \frac{1}{2}(\sigma)^2 + \theta] \end{cases}$ i.e.  $e_v = \mathcal{O}(1)$  and  $e_a = \mathcal{O}(h)$ if  $p_{k+1} > 0$  $\begin{cases} e_v = -hf[1 - \sigma - e\sigma] \\ e_q = -h((1 + e)(1 - \theta - \sigma))v_k - fh^2(e(1 - \sigma)\sigma + \frac{1}{2}(1 - \sigma)^2 - \frac{1}{2}(\sigma)^2) \end{cases}$ i.e.  $e_{v} = \mathcal{O}(h)$  and  $e_{\sigma} = \mathcal{O}(h)$ (126)

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Local error estimates for the Moreau's Time-stepping scheme

## Local error estimates for the Moreau's time-stepping

## Example (The bouncing ball (continued))

Near the finite accumulation of impact at time t = 3. Let us consider a time step such that  $[t_k, t_{k+1}] = [3 - h, 3 + h]$  and  $n_0$  such that  $h \in [1/2^{n_0}, 1/2^{n_0-1}]$ . The local error in velocity is given if the impact is detected  $p_{k+1} > 0$  by

$$e_{v} = v(3+h) - v_{k+1} = -2h - \frac{3}{2^{n_0}}.$$
 (126)

As  $h \to 0$ , we have  $n_0 \to \infty$ , and  $\frac{1}{2^{n_0}} = \mathcal{O}(h)$  and then  $e_v = \mathcal{O}(h)$ .

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Local error estimates for the Moreau's Time-stepping scheme

# Local error estimates for the Moreau's time-stepping

## To summarize

- ▶ In any case, we have O(h) in the error in coordinates and it cannot be improved if a jump occurs.
- ▶ The local error in velocity is at least  $e_v = O(1)$  if the impact is not well–forecast. In practice, this situation is usual. It illustrates the possible convergence problem that we can have in uniform norm
- Finite accumulation The order of the time-integration should be at least 0. Idea of the proof : use the fact that the velocity vanishes and is of bounded variations

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Adaptive time-step strategies

## Practical error estimates for the Moreau's time-stepping

#### Order "0" case

Standard error estimates do not apply for Order 0.

We propose to extend it to the order 0 of consistency by assuming that the the local error estimate is given by

$$e_{1/2} = 2(x_{1/2} - x_1) + \mathcal{O}(h^2)$$
(127)

where  $x_1$  is the result of the time integration with one time-step of length h and  $x_{1/2}$  with two time-steps of length h/2.

The adaptive time-step control used for smooth ODE is then apply directlyHairer et al. (1993).

Adaptive time-step strategies

#### Order "0" time-step adjustment for the Moreau's time-stepping

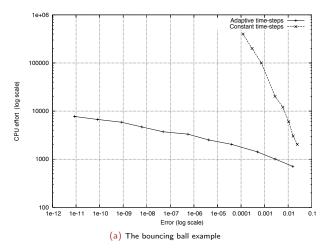


Figure: Precision Work diagram for the Moreau's time-stepping scheme. Order 0

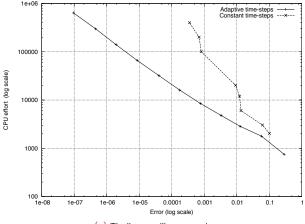
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Adaptive time-step strategies

#### Order "0" time-step adjustment for the Moreau's time-stepping



(a) The linear oscillator example

Figure: Precision Work diagram for the Moreau's time-stepping scheme. Order 0

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Adaptive time-step strategies

## Order "1" time-step adjustment for the Moreau's time-stepping

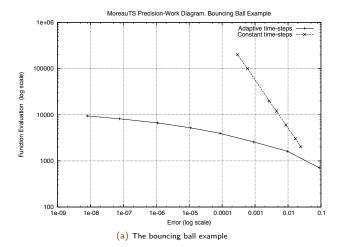


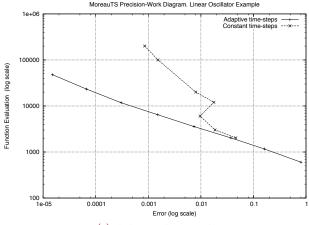
Figure: Precision Work diagram for the Moreau's time-stepping scheme. Order 1

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Adaptive time-step strategies

## Order "1" time-step adjustment for the Moreau's time-stepping



(a) The linear oscillator example

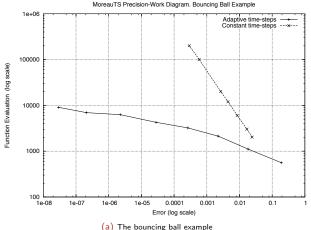


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Adaptive time-step strategies

## Order "2" time-step adjustment for the Moreau's time-stepping



(a) The bouncing ball example

Figure: Precision Work diagram for the Moreau's time-stepping scheme. Order 2

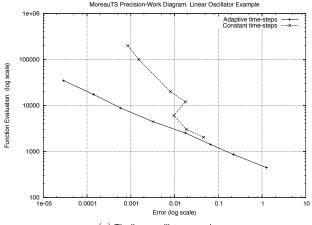
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Adaptive time-step strategies

## Order "2" time-step adjustment for the Moreau's time-stepping



(a) The linear oscillator example

Figure: Precision Work diagram for the Moreau's time-stepping scheme. Order 2

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A control based on violation

# Sizing the error in the violation of constraints

The violation of constraints is sized by the following rule:

$$e_{\text{violation}} = \|\min(0, g(q)) \circ invtol\|_{\infty}$$
(128)

Assuming that the scheme is of order 1 almost everywhere in smooth phase and may be controlled by  $e_{\rm violation}$  when an nonsmooth vent occurs, the step size adjustment is implemented by the means of the following error estimation

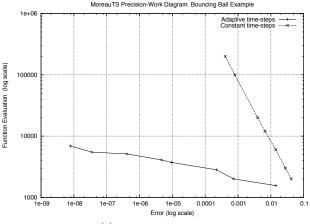
$$\operatorname{error} = \max(e_{\operatorname{violation}}, \|e_k \circ invtol\|_{\infty})$$
(129)

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A control based on violation

#### Results on two academic test examples



(a) The bouncing ball example

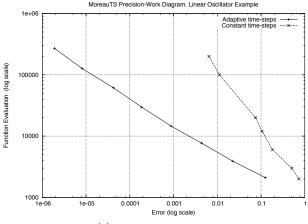
Figure: Precision Work diagram for the Moreau's time-stepping scheme. Order 0 + violation error

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A control based on violation

#### Results on two academic test examples



(a) The linear oscillator example

Figure: Precision Work diagram for the Moreau's time-stepping scheme. Order 0 + violation error

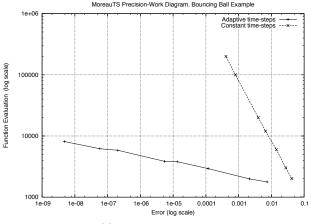
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Adaptive schemes

A control based on violation

#### Results on two academic test examples



(a) The bouncing ball example

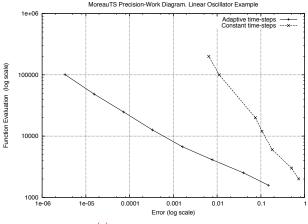
Figure: Precision Work diagram for the Moreau's time-stepping scheme. Order 1 + violation error

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A control based on violation

#### Results on two academic test examples



(a) The linear oscillator example

Figure: Precision Work diagram for the Moreau's time-stepping scheme. Order 1 + violation error

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└─ Variable order approach

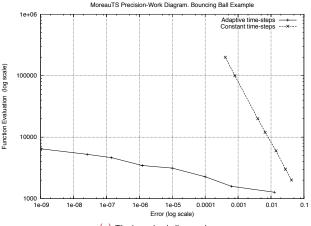
## Variable order approach. Principle

Guess the order of consistency of the integration at each step. Adapt the practical error estimation

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Variable order approach

#### Results on two academic test examples



(a) The bouncing ball example

Figure: Precision Work diagram for the Moreau's time-stepping scheme.

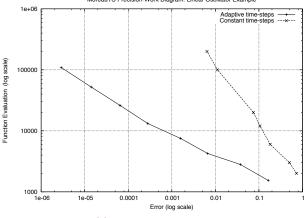
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Variable order approach

#### Results on two academic test examples



MoreauTS Precision-Work Diagram. Linear Oscillator Example

(a) The linear oscillator example

Figure: Precision Work diagram for the Moreau's time-stepping scheme.

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Adaptive schemes

└─ Variable order approach

#### Time-stepping schemes of any order

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## First attempt

In Studer et al. (2008) ; Studer (2009) the first attempt to increase the efficiency of Moreau's scheme by an extrapolation method has been published.

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# Higher Order Time-stepping schemes

### Background

Work of Mannshardt (1978) on time-integration schemes of any order for ODE/DAEs with discontinuities (with tranversality assumption)

#### Principle

- ▶ Let us assume only one event per time-step at instants t<sub>\*</sub>.
- Choose any ODE/DAE solvers of order p
- Perform a rough location of the event inside the time step of length hFind an interval  $[t_a, t_b]$  such that

$$t_* \in [t_a, t_b] \text{ and } |t_b - t_a| = Ch^{p+1} + O(h^{p+2})$$
 (130)

Dichotomy, Newton, Local Interpolants, Dense output,...

- Perform an integration on  $[t_k, t_a]$  with the ODE solver of order p
- Perform an integration on  $[t_a, t_b]$  with Moreau's time-stepping scheme
- Perform an integration on  $[t_b, t_{k+1}]$  with the ODE solver of order p

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## Integration of the smooth dynamics

Mainly for the sake of simplicity, the numerical integration over a smooth period is made with a Runge–Kutta (RK) method on the following index-1 DAE,

$$\begin{cases} M(q(t))\dot{v}(t) = F(t, q(t), v(t)) + G(q)\lambda(t), \\ \dot{q}(t) = v(t), \\ \gamma(t) = G(q(t))\dot{v}(t) = 0. \end{cases}$$
(131)

In practice, the time-integration is performed for the following system

$$\begin{cases} M(q(t))\dot{v}(t) = F(t,q(t),v(t)) + G(q)\lambda(t), \\ \dot{q}(t) = v(t), \\ 0 \leq \gamma(t) = G(q(t))\dot{v}(t) \perp \lambda(t) \geq 0 \end{cases}$$
(132)

on the time-interval I where the index set  $\mathcal{I}(t)$  of active constraints is assumed to be constant on I and  $\lambda(t) > 0$  for all  $t \in I$ .

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#### Integration of the smooth dynamics

Using the standard notation for the RK methods (see Hairer et al. (1993) for details), the complementarity problem that we have to solve at each time-step reads

$$\begin{cases} t_{ki} = t_k + c_i h, \\ v_{k+1} = v_k + h \sum_{i=1}^{s} b_i V'_{ki}, \\ q_{k+1} = q_k + h \sum_{i=1}^{s} b_i V_{ki}, \\ V'_{ki} = M^{-1}(Q_{ki}) [F(t_{ki}, Q_{ki}, V_{ki}) + G(Q_{ki})\lambda_{ki}], \\ V_{ki} = v_k + h \sum_{j=1}^{s} a_{ij} V'_{nj}, \\ Q_{ki} = q_k + h \sum_{j=1}^{s} a_{ij} V_{nj}, \\ 0 \leqslant \gamma_{ki} = G(Q_{ki}) V'_{ki} \perp \lambda_{ki} \ge 0. \end{cases}$$
(133)

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#### Assumption 3

Let I a smooth period time-interval. We assume that

1. the local order of the RK method (133) is p that is

$$e_q = e_v = \mathcal{O}(h^{p+1}) \tag{134}$$

2. starting from inconsistent initial value  $\tilde{q}_k$  such that  $\tilde{q}_k - q_k = \mathcal{O}(h^{p+1})$ , the error made by the RK method (133) is

$$\tilde{q}_{k+1} - q_{k+1} = \mathcal{O}(h^{p+1})$$
 (135)

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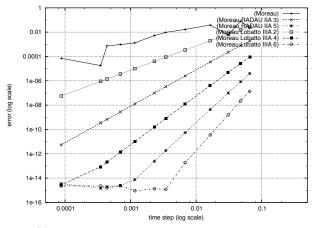
#### Theorem

Let us assume that Assumptions 1, 2 and 3 hold. The local error of consistency of the scheme is of order p in the generalized coordinates that is

$$e_q = \mathcal{O}(h^{p+1}). \tag{136}$$

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#### Results on the linear oscillator



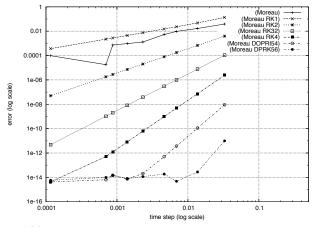
(a) The linear oscillator example with implicit Runge Kutta Method

# Figure: Precision Work diagram for the Moreau's time-stepping scheme coupled with Runge-Kutta method.

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#### Results on the linear oscillator



(a) The linear oscillator example with half explicit Runge Kutta Method

# Figure: Precision Work diagram for the Moreau's time-stepping scheme coupled with Runge–Kutta method.

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# Higher Order Time-stepping schemes

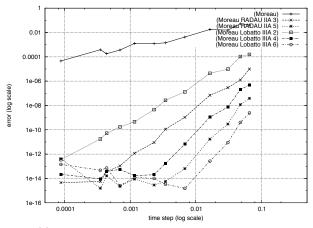
## Finite accumulation

- Repeat the whole process on the remaining part of the interval  $[t_b, t_k]$
- ▶ By induction, repeat this process up to the end of the original time step.

Acary (2009)

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#### Results on the Bouncing Ball



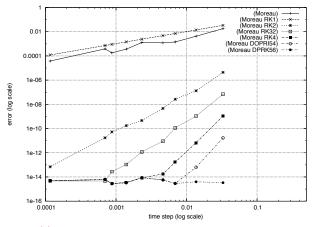
(a) The Bouncing Ball example with implicit Runge Kutta Method

Figure: Precision Work diagram for the Moreau's time-stepping scheme.

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#### Results on the Bouncing Ball



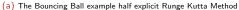


Figure: Precision Work diagram for the Moreau's time-stepping scheme.

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#### Splitting based Schemes

Numerical methods for nonsmooth mechanical systems
Splitting based Schemes
Principle

# Splitting-based methods.

#### Principle for smooth ODEs

Let us consider a smooth ODE which can be written as

$$\dot{x}(t) = f(x, t) + g(x, t)$$
 (137)

A example of splitting-based method is given by the following procedure

1. Perform the integration of f on  $[t_k, t_{k+1}]$  to obtain  $\tilde{x}(t_{k+1})$  that is

$$\tilde{x}(t_{k+1}) = x(t_k) + \int_{t_k}^{t_{k+1}} f(x, t) dt$$
(138)

2. Perform the integration of g on  $[t_k, t_{k+1}]$  with initial value  $\tilde{x}(t_{k+1})$  to obtain  $\hat{x}(t_{k+1})$  that is

$$\hat{x}(t_{k+1}) = \tilde{x}(t_{k+1}) + \int_{t_k}^{t_{k+1}} g(x,t) dt$$
(139)

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#### Properties

►  $x(t_k + 1) \neq \hat{x}(t_{k+1})$  is the general case. (except special linear case, constant dynamics, ...)

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# Splitting-based methods.

Splitting-based for Moreau scheme without continuous contact forces

The first part is

$$\begin{cases} M(q)\dot{v} = F(t, q, v), \\ \dot{q} = v, \\ q(t_k) = q_k, \quad v(t_k) = v_k \end{cases}$$
(140)

yielding to the approximations  $q_1 = q(t_{k+1})$  and  $v_1 = v(t_{k+1})$  which can integrated by any smooth ODE solvers.

The second one is given by

$$\begin{cases} M(q)\dot{v} = G(q)\lambda, \\ \dot{q} = 0, \\ y = g(q) \\ -\lambda \in \partial \psi_{\mathcal{T}_{\mathrm{IR}_{+}}(y)}(\dot{y}(t^{+}) + e\dot{y}(t^{-})) \\ q(t_{k}) = q_{1}; v(t_{k}) = v_{1}; \end{cases}$$
(141)

and leads to the approximation  $q_{k+1} = q(t_{k+1})$  and  $q_{k+1} = q(t_{k+1})$ .

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#### Splitting-based methods with constants time-step.

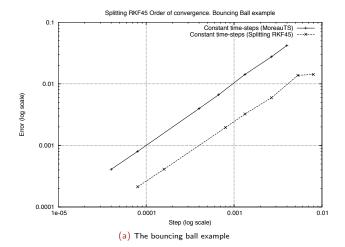


Figure: Empirical order of convergence of the Splitting RKF45 time-stepping scheme

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#### Splitting-based methods with constants time-step.

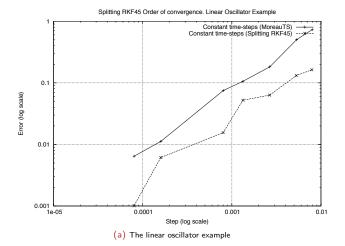


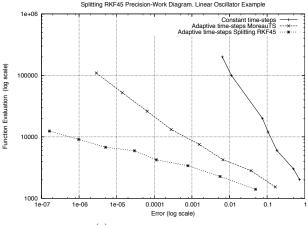
Figure: Empirical order of convergence of the Splitting RKF45 time-stepping scheme

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#### Splitting-based methods with adaptive time-step.



(a) The linear oscillator example

Figure: Empirical order of convergence of the Splitting RKF45 time-stepping scheme

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# Splitting-based methods.

## Splitting-based for Moreau scheme with continuous contact forces

The first part is

$$\begin{cases}
M(q)\dot{v} = F(t, q, v) + r(t), \\
\dot{q} = v, \\
y = g(q) \\
-r(t) \in \partial \psi_{T_{\mathbb{R}_{+}}(y)}(\dot{y}(t)) \\
q(t_{k}) = q_{k}, \quad v(t_{k}) = v_{k}
\end{cases}$$
(142)

yielding to the approximations  $q_1 = q(t_{k+1})$  and  $v_1 = v(t_{k+1})$  which can integrated by any smooth ODE solvers.

The second one is given by

$$\begin{cases}
M(q)\dot{v} = G(q)\lambda, \\
\dot{q} = 0, \\
y = g(q) \\
-\lambda \in \partial \psi_{T_{\mathrm{IR}+}(y)}(\dot{y}(t^{+}) + e\dot{y}(t^{-})) \\
q(t_{k}) = q_{1}; v(t_{k}) = v_{1};
\end{cases}$$
(143)

and leads to the approximation  $q_{k+1} = q(t_{k+1})$  and  $q_{k+1} = q(t_{k+1})$ .

## Time-discontinuous Galerkin Method

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└─ Time–discontinuous Galerkin Method └─ Principle

# Principle

Schindler and Acary (2011)

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Time-discontinuous Galerkin Method

Principle

#### Objectives

The smooth dynamics and the impact equations

Reformulations of the unilateral constraints on Different kinematics levels

Reformulations of the smooth dynamics at acceleration level.

The case of a single contact.

The multi-contact case and the index-sets

Comments and extensions

#### Event-tracking schemes

Time Discretization of the nonsmooth dynamics Time Discretization of the kinematics relations Discretization of the unilateral constraints Summary Moreau's time-stepping Schatzman-Paoli's scheme Empirical order Time-stepping schemes Comparison Newmark's scheme.

HHT scheme Generalized  $\alpha$ -methods

Newmark-type schemes for flexible multibody systems

Time-continuous energy balance equations Energy analysis for Moreau–Jean scheme Energy Analysis for the Newmark scheme Smooth ODE time integration

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- V. Acary. Toward higher order event-capturing schemes and adaptive time-step strategies for nonsmooth multibody systems. Research Report RR-7151, INRIA, 2009. URL http://hal.inria.fr/inria-00440771/en.
- P. Ballard. The dynamics of discrete mechanical systems with perfect unilateral constraints. *Archives for Rational Mechanics and Analysis*, 154:199–274, 2000.
- J. Chung and G.M. Hulbert. A time integration algorithm for structural dynamics with improved numerical dissipation: the generalized-α method. *Journal of Applied Mechanics, Transactions of A.S.M.E*, 60:371–375, 1993.
- M. Géradin and D. Rixen. *Théorie des vibrations. Application à la dynamique des structures.* Masson, Paris, 1993.
- C. Glocker. Set-Valued Force Laws: Dynamics of Non-Smooth systems, volume 1 of Lecture Notes in Applied Mechanics. Springer Verlag, 2001.
- E. Hairer, S.P. Norsett, and G. Wanner. *Solving Ordinary Differential Equations I. Nonstiff Problems.*, volume 8 of *Series in Comput. Mathematics.* Springer, second revised edition, 1993.
- H.M. Hilber, T.J.R. Hughes, and R.L. Taylor. Improved numerical dissipation for the time integration algorithms in structural dynamics. *Earthquake Engineering Structural Dynamics*, 5:283–292, 1977.
- T.J.R. Hughes. The Finite Element Method, Linear Static and Dynamic Finite Element Analysis. Prentice-Hall, New Jersey, 1987.
- R. Mannshardt. One-step methods of any order for ordinary differential equations with discontinuous right-hand sides. *Numerische Mathematik*, 31:131–152, 1978.

Time-discontinuous Galerkin Method
Principle

- J. J. Moreau. Approximation en graphe d'une évolution discontinue. *RAIRO, Anal. Numr.*, 12:75–84, 1978.
- N.M. Newmark. A method of computation for structural dynamics. Journal of Engineering Mechanics, 85(EM3):67–94, 1959.
- T. Schindler and V. Acary. Timestepping schemes for nonsmooth dynamics based on discontinuous Galerkin methods: definition and outlook. Research Report RR-7625, INRIA, May 2011. URL http://hal.inria.fr/inria-00595460/en/.
- C. Studer. Numerics of Unilateral Contacts and Friction. Modeling and Numerical Time Integration in Non-Smooth Dynamics, volume 47 of Lecture Notes in Applied and Computational Mechanics. Springer Verlag, 2009.
- C. Studer, R. I. Leine, and Ch. Glocker. Step size adjustment and extrapolation for time stepping schemes in non-smooth dynamics. *International Journal for Numerical Methods in Engineering*, 76(11):1747–1781, 2008.

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